# 고속 열차 터널의 공기압력 감소를 위한 압력 제어 시스템 

서상연, 하희상*, 이상필
지에스건설(주) 기술본부

# Effect of a Pressure Relief System in a High-speed Railway Tunnel 

Sang Yeon Seo, Heesang Ha*, and Sang Pil Lee<br>GS E\&C

*Comesponding author: hsha@gsenc.com

Received: June 5, 2018
Revised: June 18, 2018
Accepted: June 26, 2018


#### Abstract

High-speed trains have been developed widely in many countries in order to transport large quantity of people and commodities rapidly. When a high speed train enters a tunnel, aerodynamic resistance is generated suddenly. The resistance caused from air pressure induces micro pressure wave and discomfort to passengers in a train. Therefore, a pressure relief system should be installed in a tunnel to reduce the resistance acting against the running train in a tunnel. Additionally, the shape of a grain should be streamlined in order to reduce aerodynamic resistance caused by a high-speed train. The cross-section of a tunnel also should be carefully designed to reduce discomfort of passengers. This study represents the effect of pressure relief ducts installed between two running tunnels. The pressure relief duct was integrated with a cross-passage in order to save cost and construction time. One-dimensional network numerical simulations were carried out in order to estimate the effect of pressure relief systems.


Keywords: Air pressure in a tunnel, One-dimensional network numerical analysis, Pressure change within time interval, Pressure relief duct, Aural discomfort

## 초록

고속 열차는 승객과 화물을 대량으로 빠른 시간에 운송할 수 있어 세계 여러 나라에서 고속철도 건설이 증가하고 있다. 열차가 고속으로 주행할 경우 열차의 전두부에 공기 저항이 발생하며, 이러한 공기 저항 을 감소시키기 위하여 열차의 형상을 유선형으로 설계한다. 고속으로 주행하는 열차가 터널에 진입할 때, 터널 내에서 발생한 공기 저항으로 인하여 개활지 주행 시 보다 훨씬 큰 동력이 요구된다. 따라서 열 차가 터널에 진입할 때 열차에 작용하는 공기 저항을 감소시키기 위하여 열차의 주행 속도를 감소시킨 다. 이렇게 열차의 속도를 감소시킬 경우, 고속 열차의 운송 능력 및 장점이 감소되기 때문에 터널 내에서 열차 주행으로 인하여 발생되는 공기 저항을 감소시키는 설비가 필수적이다. 이 연구에서는 터널 내에서 열차의 고속 주행을 위하여 필요한 공기 압력 제어 시스템의 효과를 분석하기 위하여 1차원 수치해석을 수행하였다. 1 차원 수치해석 프로그램을 통하여, 터널의 단면적 및 공기압력 제어 덕트의 단면적과 배치 간격이 터널 내에서 발생하는 공기 저항에 미치는 영향을 상세히 분석하였다.

핵심어: 터널 내 공기 압력, 1 차원 수치해석, 시간별 압력 변화량, 압력 제어 덕트, 이명감

## 1. Introduction

Recently, investment for high-speed railways has been increased due to large demand on fast and secure public transportation. In addition, underground transport system, such as a high-speed train has become faster than in the past in order to satisfy passengers' expectations. Therefore, many construction plans for high-speed railway have been planned in many countries. South Korea also has constructed high-speed underground transportation system in a metropolitan area. The system is called as a Great Train Express (GTX) and planned to use a high-speed train that can travel with the maximum speed of $350 \mathrm{~km} / \mathrm{h}$. High-speed trains need a track with larger curvature than low-speed trains in order to maintain its speed. Therefore, tunnel is one of the most important infrastructures for high-speed trains.

According to Barthes et al. (1994), when a high-speed train enters a tunnel, large aerodynamic resistance is generated. As the speed of a train increases, the effects of aerodynamics become important on the design of tunnels and trains. Complicated distribution of aerodynamic pressure occurs in front of a running train. The nose of a train has a streamlined shape to decrease aerodynamic resistance acting on a train (Fairbairn, 1995; Vardy, 1996). However, this aerodynamic pressure on a train becomes much more complicated when a train enters a tunnel. This complicated aerodynamic pressure by a running train in a tunnel causes problems such as micro pressure wave and aural discomfort on passengers. Pressure comfort and health protection of passengers should be guaranteed according to design codes and specifications (UIC 660, 2002). UIC 779 presents that the maximum pressure difference on passengers in a train should be less that 10 kPa (UIC 779, 2005). Additionally, a non-pressure tight train should have a pressure difference less than 4.5 kPa for 4 seconds in a double-track railway tunnel, and 3.0 kPa for 4 seconds in a single-track tunnel. For a pressure tight train, a pressure difference should be lower than 1.0 kPa in a second, 1.6 kPa for 4 seconds, and 2.0 kPa for 10 seconds in either single-track or double-track tunnel (UIC 660, 2002; UIC 779, 2005). In South Korean standard for a railway tunnel, pressure difference in a tunnel should be lower than 800 Pa for 3 seconds when there is only one train running in a tunnel and $1,250 \mathrm{~Pa}$ for 3 seconds if another train passes through in the opposite tunnel.

Atkins (1999) showed that aerodynamic resistance developed by a train in a tunnel has a significant influence on not only aural comfort, but also traction demand required for a train passage in a tunnel. According to Henson (1995), when a high-speed locomotive enters a tunnel, large traction power demand is necessary due to aerodynamic pressure resistance in a tunnel. Reinke (2011) proposed that the speed of train is usually decreased before the train enters a tunnel, so various measures are adopted in a tunnel not to lower the speed of a locomotive. For example, the Channel Tunnel connecting United Kingdom and France adopted circular ducts with a diameter of 2 m every 250 m between two running tunnels (Fig. 1). These pressure relief ducts could alleviate air pressure resistance in the Channel Tunnel and let Eurostar trains travel with the speed of $160 \mathrm{~km} / \mathrm{h}$ (Southwood, 1994). These ducts are arch shaped in order to prevent interference with running tunnels (Southwood, 1994). Various numerical simulations were carried out to investigate and analyze the effect of a pressure relief system in this study. The maximum pressure difference caused by a running train in a tunnel was thoroughly investigate with various scenarios.


Fig. 1. Pressure relief ducts installed in the Channel tunnel

## 2. Aerodynamic resistance in a tunnel

Abrupt aerodynamic resistance might be developed in front of a train when it enters a tunnel (Reinke, 2011). A train entering a tunnel displaces air and the air pressure increases abruptly at the tunnel portal at the train nose. The overpressure leads to some air flowing back alongside the train and out of the entrance of the tunnel. The remainder passes down the tunnel behind a pressure wave front. The pressure wave propagates with the speed of sound as a compression wave along the tunnel (Fig. 2).


Fig. 2. Pressure deviation caused by a train along the tunnel(1)(Reinke, 2011)

As the tail of the train enters the tunnel, a sudden pressure drop occurs behind the train. This second pressure wave also propagates with the speed of sound as a decompression wave along the tunnel. This aerodynamic resistance, or aerodynamic drag leads to a characteristic pressure distribution along the train. As the nose of a train reaches a certain point in the tunnel, a pressure drop occurs and the pressure further drops due to the longitudinal friction along the train surface. Behind the train tail the pressure increases again as shown in Fig. 3. The pattern of the pressure distribution in the tunnel continuously changes (Fig. 4).

When the wave is reached the exit portal, it is partially reflected and travels back as an expansion wave (Reinke and

Busslinger, 2011). Ends of tunnels reflect compression waves as expansion waves, and expansion waves as compression waves. An expansion wave takes in air at the end of the tunnel, and it propagates as a compression wave. Further reflections at portals, changes of free cross-sectional area of tunnels and trains might lead to various compression and expansion waves oscillating simultaneously in the tunnel. While oscillating in the tunnel, the amplitude of the waves decreases due to friction and losses due to reflection.


Fig. 3. Pressure deviation caused by a train along the tunnel(2)(Reinke, 2011)


Fig. 4. Pressure deviation caused by a train along the tunnel(3)(Reinke, 2011)

## 3. Design codes for pressure deviation in a tunnel

For high speed railway tunnels, pressure comfort is one of the most important aspect to determine the size and shape of the cross-sectional area of a tunnel. The criteria for pressure comfort are commonly defined by the maximum pressure changes within a given time period. Many countries propose design codes to determine the cross-sectional area of tunnels in terms of aural comfort to passengers. The International Union of Railway(UIC) harmonized different criteria and proposed two sets of pressure comfort criteria, such as UIC 660 and UIC 779-11 (Table 1). UIC 779-11 proposes that the pressure change should be lower than 1.0 kPa for 1 second and 2.0 kPa for 10 seconds. However, the pressure change should be lower than 0.5 kPa for 1 second and 1.0 kPa for 10 seconds in UIC 660 . This high comfort level according UIC-Code 660 was initially proposed by German Rail and implemented in their initial high-speed network.

The criteria were developed on the premise of "not to have complaints" about pressure comfort and to be on the safe side.

In tunnels with strong gradients it turned out, however, that it was impractical to respect the 60 -s-time-interval as the change in height leads to a significant pressure change already. Therefore, only the time intervals of 1,3 and 10 s are considered. South Korea proposes pressure comfort criteria similar to UIC 660 (Table 1) and additional pressure comfort criteria for the cases when there is only one running train or two crossing trains in a tunnel (UIC 660, 2002; UIC779, 2005).

Table 1. Maximum pressure change $(\mathrm{kPa})$ within a time interval

| Time interval <br> (seconds) | UIC 660 | UIC 770-11 | Korean Code |
| :---: | :---: | :---: | :---: |
| 1 | 0.5 | 1.0 | 0.5 |
| 3 | 0.8 | - | 0.8 (one train) |
| 4 | - | 1.6 | 1.25 (two trains) |
| 10 | 1.0 | 2.0 | - |
| 60 | 2.0 | - | 1.0 |

## 4. Numerical simulation

### 4.1. Model and conditions for numerical simulation

Numerical simulations were carried out in order to analyse the pressure changes in time period in a tunnel. Details of a numerical model shown in Fig. 5 was based on the preliminary design of Honam-Jeju subsea tunnel in Korea. The tunnel in the preliminary design is a mechanically driven tunnel by TBM and has a circular cross section with area of $57.25 \mathrm{~m}^{2}$. The tunnel is a single-tube and double track tunnel and pressure relief ducts are installed between two tunnels (Fig. 5). This pressure relief duct is incorporated to decrease aerodynamic resistance in a tunnel because it is very hard to construct a vertical shaft in a subsea tunnel. The length of the tunnel in numerical simulation is 35 km and the area of the pressure relief duct and distance between ducts are assumed variously for different simulation cases. Table 2 shows parameters used in THERMOTUN, which is a numerical simulation program being widely used for design of a tunnel. THERMOTUN is developed in Dundee Tunnel Research (DTR) to analyse pressure changes in a tunnel using one-dimensional network models. The program has been widely used for many projects and validated through a lot of researches (Vardy and Dayman, 1979).


Fig. 5. Schematic diagram for numerical modelling

Table 2. Parameters for numerical simulation

|  | Parameter | Value |
| :---: | :---: | :---: |
| Train | Shape of head | Hypothetical |
|  | Length | $\mathrm{L}_{\text {train }}=201 \mathrm{~m}$ |
|  | Cross-sectional area | $\mathrm{A}_{\text {train }}=9.34 \mathrm{~m}^{2}$ |
|  | Perimeter | $\mathrm{P}_{\text {train }}=11.67 \mathrm{~m}$ |
|  | Maximum speed | $\mathrm{V}_{\text {train }}=350 \mathrm{~km} / \mathrm{h}$ |
|  | Time constant of pressure tightness | $\tau=18 \mathrm{sec}$ |
|  | Longitudinal friction factor | $\mathrm{f}_{\text {train }}=0.003$ |
|  | Nose and tail loss coefficient | $\mathrm{k}_{\text {nose }}=0.05, \mathrm{k}_{\text {tail }}=0.07$ |
| Tunnel | Length | $\mathrm{L}_{\text {tun }}=35,000 \mathrm{~m}$ |
|  | Type | Twin tube, single track |
|  | Cross-sectional area | $\mathrm{A}_{\text {tun }}=60 \mathrm{~m}^{2}$ |
|  | Perimeter | $\mathrm{P}_{\text {tun }}=31.5 \mathrm{~m}$ |
|  | Tunnel friction factor (D'arcy-Weisbach definition) | $\lambda_{\text {tun }}=0.024$ |
|  | Tunnel friction factor | $\mathrm{f}_{\text {tun }}=0.06$ |
|  | Length of pressure relief duct | $\mathrm{L}_{\mathrm{rd}}=30 \mathrm{~m}$ |
|  | Inclination | $\mathrm{i}=0 \%$ |
|  | Ambient, tunnel temperature | $15^{\circ} \mathrm{C}$ |
|  | Normal pressure at sea level | $101,300 \mathrm{~Pa}$ |

In order to investigate the effect of a pressure relief duct on pressure changes in a tunnel, 41 simulation cases were prepared. Each simulation case has a unique combination of various parameters, such as the speed of train, the cross-sectional area of a tunnel and a pressure relief duct, and the spacing between ducts as shown in Table 3.

The train is assumed to run with the speed (Vtrain) of either $300 \mathrm{~km} / \mathrm{h}$ or $350 \mathrm{~km} / \mathrm{h}$, because the effect of pressure relief system becomes significant in high speed railway tunnels. Currently, high speed trains in Korea, KTX, runs with the speed of $300 \mathrm{~km} / \mathrm{h}$ in open tracks. The cross-sectional areas of tunnels $\left(\mathrm{A}_{\text {tunnel }}\right)$ are $42.59 \mathrm{~m}^{2}$ and $57.25 \mathrm{~m}^{2}$. The former is the area of the Channel Tunnel and the latter is from the area of the preliminary design of Honam-Jeju subsea tunnel. The area of the cross-section of a pressure relief duct ( $\mathrm{A}_{\text {duct }}$ ) varies from $0.70 \mathrm{~m}^{2}$ to $3.14 \mathrm{~m}^{2}$. The smallest value is from the preliminary design of Honam-Jeju subsea tunnel and the $3.14 \mathrm{~m}^{2}$ is the area of the pressure relief duct installed in the Channel Tunnel. The distance between pressure relief ducts ( $\mathrm{S}_{\text {duct }}$ ) varies from 250 m to 300 m to investigate the effect of pressure relief ducts on the pressure change in a tunnel. Pressure tightness of a train $(\tau)$ was set to 18 s because KTX high-speed train is manufactured with pressure tightness larger than 18 s .

### 4.2 Analysis results

Aerodynamic pressure occurred in a tunnel fluctuates as a train passes through the tunnel. Fig. 6 shows how air pressures acting on the nose, middle, and tail of a train are changed for 1 second and 10 seconds. The result in Fig. 6 is from the

Table 3. Parameters used in numerical simulations and results

| Simulation case | Vtrain <br> (km/h) | Atunnel $\left(\mathrm{m}^{2}\right)$ | Aduct <br> ( $\mathrm{m}^{2}$ ) | Sduct <br> (m) | Pressure change in time period of $1 \mathrm{~s}(\mathrm{kPa}$ in s$)$ | Pressure change in time period of $10 \mathrm{~s}(\mathrm{kPa}$ in s$)$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 350 | 42.59 | 0.70 | 250 | 0.284 | 2.071 |
| 2 | 350 | 42.59 | 0.70 | 275 | 0.286 | 2.086 |
| 3 | 350 | 42.59 | 0.70 | 300 | 0.288 | 2.098 |
| 4 | 350 | 42.59 | 1.00 | 250 | 0.276 | 2.013 |
| 5 | 350 | 42.59 | 1.00 | 275 | 0.278 | 2.027 |
| 6 | 350 | 42.59 | 1.00 | 300 | 0.279 | 2.038 |
| 7 | 350 | 42.59 | 2.00 | 250 | 0.262 | 1.917 |
| 8 | 350 | 42.59 | 2.00 | 275 | 0.264 | 1.929 |
| 9 | 350 | 42.59 | 2.00 | 300 | 0.265 | 1.938 |
| 10 | 350 | 42.59 | 3.14 | 250 | 0.255 | 1.860 |
| 11 | 350 | 42.59 | 3.14 | 275 | 0.256 | 1.873 |
| 12 | 350 | 42.59 | 3.14 | 300 | 0.257 | 1.883 |
| 13 | 350 | 57.25 | 0.70 | 250 | 0.193 | 1.421 |
| 14 | 350 | 57.25 | 0.70 | 275 | 0.194 | 1.430 |
| 15 | 350 | 57.25 | 0.70 | 300 | 0.195 | 1.437 |
| 16 | 350 | 57.25 | 1.00 | 250 | 0.188 | 1.386 |
| 17 | 350 | 57.25 | 1.00 | 275 | 0.189 | 1.394 |
| 18 | 350 | 57.25 | 1.00 | 300 | 0.190 | 1.401 |
| 19 | 350 | 57.25 | 2.00 | 250 | 0.180 | 1.326 |
| 20 | 350 | 57.25 | 2.00 | 275 | 0.181 | 1.333 |
| 21 | 350 | 57.25 | 2.00 | 300 | 0.181 | 1.340 |
| 22 | 350 | 57.25 | 3.14 | 250 | 0.175 | 1.290 |
| 23 | 350 | 57.25 | 3.14 | 275 | 0.176 | 1.296 |
| 24 | 350 | 57.25 | 3.14 | 300 | 0.176 | 1.304 |
| 25 | 300 | 42.59 | 0.70 | 250 | 0.176 | 1.490 |
| 26 | 300 | 42.59 | 0.70 | 275 | 0.203 | 1.501 |
| 27 | 300 | 42.59 | 0.70 | 300 | 0.206 | 1.512 |
| 28 | 300 | 42.59 | 2.00 | 250 | 0.187 | 1.376 |
| 29 | 300 | 42.59 | 2.00 | 275 | 0.189 | 1.385 |
| 30 | 300 | 42.59 | 2.00 | 300 | 0.190 | 1.394 |
| 31 | 300 | 57.25 | 0.70 | 250 | 0.139 | 0.984 |
| 32 | 300 | 57.25 | 0.70 | 275 | 0.140 | 0.990 |
| 33 | 300 | 57.25 | 0.70 | 300 | 0.141 | 0.997 |
| 34 | 300 | 57.25 | 1.00 | 250 | 0.136 | 0.960 |
| 35 | 300 | 57.25 | 1.00 | 275 | 0.137 | 0.965 |
| 36 | 300 | 57.25 | 1.00 | 300 | 0.138 | 0.972 |
| 37 | 300 | 57.25 | 2.00 | 250 | 0.130 | 0.917 |
| 38 | 300 | 57.25 | 2.00 | 275 | 0.131 | 0.922 |
| 39 | 300 | 57.25 | 2.00 | 300 | 0.131 | 0.927 |
| 40 | 300 | 57.25 | 3.14 | 250 | 0.126 | 0.890 |
| 41 | 300 | 57.25 | 3.14 | 275 | 0.127 | 0.895 |

simulation case 1 , which has a train velocity of $350 \mathrm{~km} / \mathrm{h}$, cross-sectional area of a tunnel of $42.59 \mathrm{~m}^{2}$, cross-sectional area of a duct of $0.70 \mathrm{~m}^{2}$, and spacing of duct of 250 m . As shown in Fig. 6, high positive normal pressure is occurred when a tunnel enters the entrance portal. This high pressure decreased quickly as the train moves in the tunnel and there is a complicated fluctuation of pressure along the tunnel. The biggest pressure change is occurred right before the train passes through the exit portal of a tunnel. Pressure changes from 41 simulation cases were listed in Table 3 and Fig. 7. In Fig. 7, red dots are values of the maximum pressure change in 10 seconds time interval and blue dots are the values in 1 second time interval. A red line and an orange line show the pressure criteria from UIC $779-11$; the red line stand for 2.0 kPa in 10 seconds time interval and the orange line stand for 1 kPa in 1 second time interval. A light blue and dark blue line are from UIC 660 ; the light blue line stand for 1 kPa in 10 seconds time interval and the dark blue line stand for 0.5 kPa in 1 second time interval. As clearly shown in Fig. 7 and Table 2, pressure changes in a tunnel are influenced by various aspects, such as the velocity of a train, the cross-sectional area of a tunnel and a pressure relief duct and spacing between ducts. Every simulation case could meet the criteria for 1 second time interval according to both UIC 779-11 and UIC 660. However, some simulation cases could not meet the criteria for 10 second time interval.


Fig. 6. Pressure change in time period of 1 s and 10 s (case 1)

From the case no. 1 to case no. 12 simulate that a high-speed train runs with the speed of $350 \mathrm{~km} / \mathrm{h}$ in a tunnel, which has the same cross-sectional area to the Channel Tunnel. These 12 simulation cases shows very high pressure changes for 10 seconds time interval through train passage. When the cross-sectional area of a pressure relief duct is $0.7 \mathrm{~m}^{2}$ (from case 1 to case 3) and $1.0 \mathrm{~m}^{2}$ (from case 4 to case 6), the maximum pressure change exceed 2.0 kPa in 10 seconds time interval (UIC $779-11$ ) and 1.0 kPa in 10 seconds (UIC 660). As the cross-sectional area of a duct is increased to $2.0 \mathrm{~m}^{2}$ and $3.14 \mathrm{~m}^{2}$ (from case 4 to case 6 ), the maximum pressure changes are smaller than 2.0 kPa , but much higher than 1.0 kPa , which is a standard for 10 seconds time interval in UIC 660. This implies that the cross-sectional area of a pressure relief duct should be larger


Fig. 7. Pressure changes in time period of 1 s and 10 s
than $2.0 \mathrm{~m}^{2}$ in order to meet the minimum criteria of UIC 779-11.
The preliminary design of Honam-Jeju subsea tunnel has a larger cross-sectional area with $57.25 \mathrm{~m}^{2}$ (from case 13 to 24 ). As the cross-sectional area of a tunnel increased from $42.59 \mathrm{~m}^{2}$ to $57.25 \mathrm{~m}^{2}$, pressure changes could be decreased around 28 $-31 \%$ and satisfy the 10 seconds time interval criterion of UIC 779-11. However, every simulation cases with the area of 57 . $25 \mathrm{~m}^{2}$ showed pressure changes larger than 1.0 kPa in 10 seconds time period. This implies that passengers in a high-speed train running $350 \mathrm{~km} / \mathrm{h}$ may suffer pressure discomfort. Therefore, it is required to adopt other measures to decrease the maximum pressure change caused by a train in a tunnel.

The speed of a train is decreased to $300 \mathrm{~km} / \mathrm{h}$ in other 18 simulation cases (from case 25 to case 42). When a train runs with $300 \mathrm{~km} / \mathrm{h}$ in a tunnel with $42.59 \mathrm{~m}^{2}$, the maximum pressure change in 10 seconds time interval decreased $28 \%$ compared to the cases with the speed of $350 \mathrm{~km} / \mathrm{h}$. However, the pressure change in 10 seconds time interval ranges from 1.376 kPa to 1.512 kPa , and they are much higher than the values from UIC 660 (from case 25 to case 30 ). Therefore, it is necessary to increase the area of a tunnel in order to make the pressure change lower than 1.0 kPa in 10 seconds time interval (UIC 660). From the simulation case 31 to 42 , a train runs with the speed of $300 \mathrm{~km} / \mathrm{h}$ in a tunnel with a cross-sectional area of $57.25 \mathrm{~m}^{2}$. When the cross-sectional area of a duct is $0.7 \mathrm{~m}^{2}$ and the spacing of ducts is 250 m , the maximum pressure change is 0.984 kPa in 10 seconds time interval and it satisfies 10 seconds standard of UIC 660 . As the spacing of ducts increases, the maximum pressure change increases slightly. In addition, the pressure change can be also decreased with increasing cross-sectional area of a pressure relief duct. Therefore, it is obvious that the area of a tunnel should be larger than $57.25 \mathrm{~m}^{2}$ and pressure relief ducts with cross-sectional area larger than $0.7 \mathrm{~m}^{2}$ should be installed with the spacing smaller than 300 m when a high-speed train runs with the maximum speed of $300 \mathrm{~km} / \mathrm{h}$ to satisfy UIC $779-11$ and UIC 660 standards.

## 5. Conclusion

A one-dimensional numerical simulation using THERMOTUN was carried out in order to select the optimal area of a tunnel and investigate the effect of pressure relief ducts installed between two running tunnels. The pressure change occurred in a tunnel is one of the most important factor for designing the cross-sectional area of a tunnel. Therefore, detailed 41 numerical analysis were conducted with various train speed, cross-sectional area of a tunnel and a pressure relief duct, and spacing between ducts. According to the results of numerical analysis, the optimum combination of a tunnel and a duct was proposed for a high-speed railway tunnel which a locomotive can run with the speed of $300 \mathrm{~km} / \mathrm{h}$. The specifications of a tunnel and a duct from this research could meet the standard for the maximum pressure change in a tunnel according to UIC 779-11 and UIC 660. The results from this study can be summarized as follows.

Pressure changes in time interval of 1 second and 10 seconds become significantly different according to the speed of a locomotive, cross-sectional area of a tunnel and a pressure relief duct, and spacing between ducts. All of 41 simulation cases could satisfy the criteria for the pressure change in 1 second time interval of both UIC 779-11 and UIC 660. There are some simulation cases which could not meet the 10 seconds criteria of UIC 660 when a train runs with $350 \mathrm{~km} / \mathrm{h}$. Even though the speed of a train is decreased to $300 \mathrm{~km} / \mathrm{h}$, the maximum pressure change in 10 seconds is higher than 1.0 kPa when the cross-sectional area of a tunnel is $42.59 \mathrm{~m}^{2}$. Therefore, it is necessary to increase the area of a tunnel to $57.25 \mathrm{~m}^{2}$ in order to lower the maximum pressure change in the tunnel.

If a high-speed locomotive runs with $350 \mathrm{~km} / \mathrm{h}$ in the Channel Tunnel, which has a cross-sectional area of $49.25 \mathrm{~m}^{2}$, the maximum pressure change becomes larger than 2 kPa in 10 seconds time interval. If a pressure relief ducts with the cross-sectional area of $3.14 \mathrm{~m}^{2}$ were installed every 250 m , the maximum pressure change in 10 seconds can be lower that the criteria of UIC 779-11. However, it highly surpasses the 10 seconds criteria of UIC 660 and passengers in a train may feel aural discomfort in a tunnel. This implies that a high-speed locomotive will not be able to run faster than $300 \mathrm{~km} / \mathrm{h}$ in the Channel Tunnel due to excessive air pressure inside a tunnel. A tunnel with larger area ( $57.25 \mathrm{~m}^{2}$ ) can have much lower pressure changes compared to the Channel Tunnel. However, if a train runs faster than $300 \mathrm{~km} / \mathrm{h}$ in a tunnel with a cross-sectional area of $57.25 \mathrm{~m}^{2}$, the maximum pressure change in 10 seconds period becomes greater than 1.0 kPa according to UIC 660 . Therefore, the speed of train should be restricted to $300 \mathrm{~km} / \mathrm{h}$ in order to meet UIC 660 criteria for pressure comfort.

## Submission of contributions

This research was supported by a grant (Project number: 18SCIP-B066321-06 (Development of Key Subsea Tunnelling Technology)) from Infrastructure and Transportation Technology Promotion Research Program funded by Ministry of Land, Infrastructure and Transport of Korean government.

## REFERENCES

Atkins W.S., 1999, Piston relief ducts for AlpTransit Gotthard, The Channel Tunnel Experience, Institution of Engineering and Technology, Conference Publication (Book 433), 20-23.
Barthes, H., Bordas, A. and Bouillot, D., 1994, Tunnels - special works, Proceedings of the In-stitution of Civil Engineers, The Institution of Civil Engineers, Engineering Channel Tunnel, Part 3: French Sectoin, 63-75.
Fairbairn, A. G., 1995, Tunnel ventilation, including aerodynamic, Proceedings of the Institution of Civil Engineers, The Institution of Civil Engineers, Engineering Channel Tunnel, Part 4: Transport Systems, 32-41.
Henson, D., 1995, Aerodynamics, Ventilation and Cooling the tunnel, Engineering the Channel Tunnel, Kirkland, J., E\&FN SPON/EUROTUNNEL, 217-220.
Reinke, P., and Busslinger, A., 2011, Improvement of aero and thermodynamics of rail tunnels by cross-connections with shut-off devices, Research 2011, HBI Haerter Ltd., Swiss
Southwood, A. J., 1994, The Channel Tunnel : A Designer's Perspective, A. J., Mott MacDonald, 11-13.
UIC 660, 2002, Measures to ensure the technical compatibility of high-speed trains, Technical document, Railway Technical Publications UIC leaflet 660.

UIC. 779, 2005, Determination of railway tunnel cross-sectional area on the basis of aerodynamic considerations, Technical document, Railway Technical PublicationsUIC leaflet 779-11.
Vardy, A.E. and Dayman B., 1979, Alleviation of tunnel entry pressure transient: 2 - theoretical modelling and experimental correlation, Proceeding of the 3rd International Symposium on the Aerodynamics and Ventilation of Vehicle Tunnels, BHRA Fluid Eng., Cranfield, 363-376.
Vardy, A.E., 1996, Aerodynamic drag on trains in tunnels, Part 1: synthesis and definitions, Pro-ceedings of the Institute of Mechanical Engineers, Part F, J. Rail Rapid Transit 210(1), 29-38.

