

# Thermal Analysis of High Speed Induction Motor by Using Lumped-Circuit Parameters

Pil-Wan Han\*, Jae-Hak Choi\*, Dong-Jun Kim\*, Yon-Do Chun\* and Deok-Je Bang†

**Abstract** – This paper deals with the electromagnetic and thermal analysis of high speed induction motor. The induction motor is analyzed by time-varying magnetic finite element method and its thermal analysis is carried out by using analytical lumped-circuit method. Analysis results are compared with the experiment of 29kW high speed motor prototype at 12,000rpm.

**Keywords:** High speed, Induction motor, Thermal analysis, Lumped parameter

## 1. Introduction

The thermal analysis of electric machine has received less attention than electromagnetic analysis. Motor designers have only superficially dealt with the thermal design aspects by specifying a limiting value of current density or some other sizing variables. As it is impossible to accurately analyze one (electromagnetic) without the other (thermal), the losses are critically dependent upon the temperature and vice versa [1].

Electric motor thermal analysis can be divided into the following two basic types: analytical lumped-circuit and numerical methods. The analytical approach has the advantage of being very fast to calculate, but the developer of the circuit model must invest effort in defining a circuit that accurately models the main heat transfer paths [2-4].

The main strength of numerical analysis is that any device geometry can be modeled. However, it is very demanding in terms of model setup and computational time. There are two types of numerical analysis: the finite-element analysis (FEA) and the computational fluid dynamics (CFD) [5, 6].

In order to obtain an accurate thermal model for an electrical machine, both the analytical and numerical approaches benefit from the previous experience of the designer. This is due to some thermal phenomena being dependent upon the manufacturing process, such as the smoothness of the outer stator lamination and the inner housing surface [2].

In this paper, to analyze thermal characteristics of high speed induction motor of 29kW-12,000 rpm, lumped-circuit parameters thermal network method of commercial software is used [4]. The Induction motor of this study is driven by sinusoidal voltage without PWM inverter.

## 2. Loss Analysis and Test of High Speed Motor

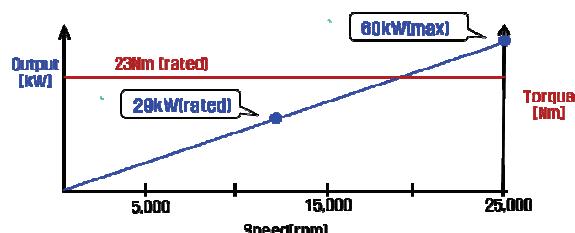
### 2.1 Specification of motor

In the last decade increasing attention has been given to high-speed drives for wider range of application, such as compressors, turbochargers, blowers, flywheel energy storages and spindles for machine tool [2].

This paper studies 29kW-12,000rpm (Max. 60kW-25,000rpm) high speed induction motor including water cooling jacket as shown in Table 1. Fig. 1 is torque and power characteristics of motor which could be used in a high speed spindle system.

**Table 1.** Specification of high speed motor

Items	Unit	Rated	Maximum
Power	kW	29	60
Rated line voltage	V	182	380
Rated frequency	Hz	400	833
Rated speed	RPM	12,000	25000
Cooling condition		Water cooling (7L/min, 20°C)	



**Fig. 1.** Rated performance vs Speed

### 2.2 Design of motor

The high speed motor of this paper is designed as two types. First model is designed on high current density and it is expected to have bad thermal characteristics. In addition, this design has problems in centrifugal casting copper rotor due to small rotor slot. The second model is modified to make up for the first model's defects as show

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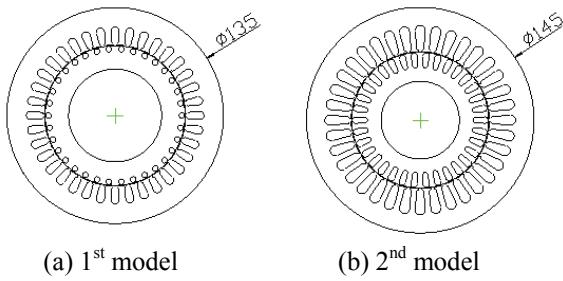


Fig. 2. Drawings of core.

Table 2. Design of High Speed Motor

Items	Unit	1 <sub>st</sub>	2 <sub>nd</sub>
Out diameter of stator	mm	135	145
Inner diameter of stator	mm	88	88
Stack length	mm	150	150
Current density of stator	A/mm <sup>2</sup>	18.0	8.44
Current density of rotor	A/mm <sup>2</sup>	20.7	7.09
Flux density of airgap	T	0.76	0.60
Electrical steel thickness	mm	0.35	0.2
Core loss at 1.5T/834hz	[W/kg]	139	139
Rotor conductor		copper	

in Table 2 and Fig. 2. Area of stator and rotor slots is increased more than two times to reduce current and flux density and improves the productivity of centrifugal casting copper rotor.

### 2.3 Loss analysis and test results

The basic losses such as copper loss and iron loss are calculated by 2D electro-magnetic FEM using sinusoidal voltage.

The induction motor is analyzed by time-varying magnetic finite element analysis at rated condition (182V/400Hz) by using sinusoidal voltage. Fig. 3 shows the flux density distribution of designed models (2<sup>nd</sup> model). Maximum value of stator yoke and teeth flux density is 1.4 [T] and 1.6 [T] respectively.

Table 3 shows the loss analysis by 2D FEM and test results in sinusoidal voltage. If this machine is driven by PWM inverter, its efficiency can decrease because of harmonic loss. A stator winding and rotor bar temperature

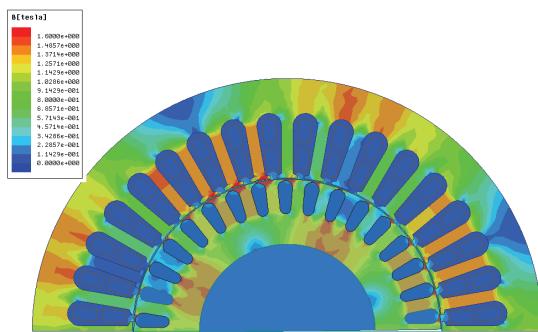


Fig. 3. Flux density distribution

Table 3. Comparison of loss analysis and test results

Method	Eff. [%]	Losses [W]					
		Total	Stator copper	Rotor copper	Iron	Mech.	Stray load
FEM	92.6	2306	535	427	429	174	743
TEST	91.9	2555	622	453	420	197	863

are set on 80°C and 150°C. A stray load loss is calculated by about 2.5% of output power and a mechanical loss by using Takeuchi's formula [11].

Testing of prototype is done by IEC 60034-2-1, loss segregation method by means of output power.

### 2.4 Prototype assembling process

Fig. 4 shows a process assembling prototype. A rotor bar is made by centrifugal copper casting and stator is molded by a synthetic resin to improve cooling capacity.

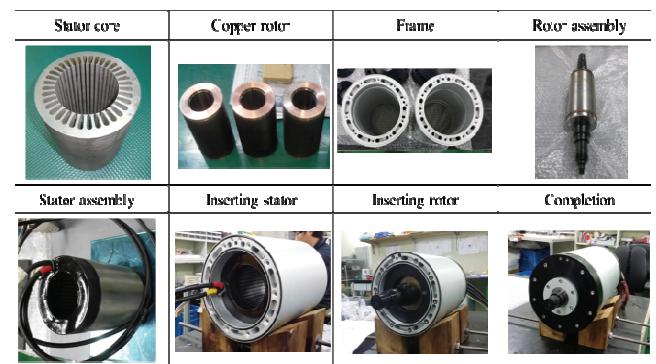


Fig. 4. Assembling prototype

## 3. Parameter Calculations

### 3.1 Lumped parameter thermal network (LPTN) method

LPTN method has been successfully used for thermal analysis of electric motors using commercial software as shown in [4]. All thermal parameters, such as conduction ( $R_c$ ), radiation ( $R_r$ ), convection thermal resistances ( $R_{cv}$ ) and flow resistance ( $R_f$ ) in case of forced convection heat transfer are calculated by program considering cooling system as shown in Eqs. (3)~(6) [1].

$$R_c = \frac{L}{kA} \quad (3)$$

$$R_r = \frac{1}{h_R A} \quad (4)$$

$$R_{cv} = \frac{1}{h_C A} \quad (5)$$

$$R_f = \frac{k\rho}{2A^2} \quad (6)$$

where  $L$  is the path length,  $A$  is the path or surface area,  $k$  is the thermal conductivity,  $h_R$  is heat-transfer coefficient,  $h_C$  is the convection-heat-transfer coefficient and  $\rho$  is the air or fluid density depending on the temperature.

### 3.2 Analysis of forced convection (water jacket)

In this study, high speed motor is cooled by water jacket as shown in Fig. 5. In a forced convection system, if the fluid velocity is large, the turbulence is induced. The magnitude of  $Re$  (Reynolds number) of Eq. (7) is used to determine if there is laminar or turbulent flow in a forced convection system [8, 12, 15~17].  $Re$  is calculated to 13,140 in this model. For fully developed turbulent flow ( $3,000 < Re < 10^6$ ),  $Nu$  (Nusselt number) and heat transfer coefficient are calculated by Eq. (9) and (10) [13].

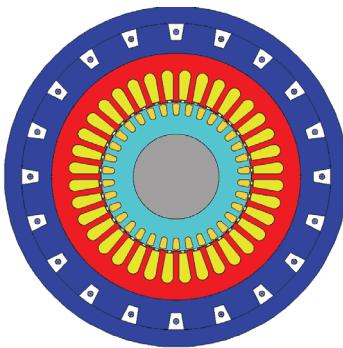
$$Re = \rho \cdot v \cdot L / \mu \quad (7)$$

$$Pr = c_p \cdot \mu / k \quad (8)$$

$$Nu = \frac{f}{8} \cdot \frac{(Re-1000) \cdot Pr}{1+12.7 \cdot (f/8)^{0.5} \cdot (Pr^{2/3}-1)} \quad (9)$$

$$Nu = h \cdot L / k \quad (10)$$

where  $Pr$  is Prandtl number,  $v$  is fluid velocity [m/s],  $c_p$  is fluid specific heat capacity [kJ/(kg·°C)],  $\mu$  is fluid dynamic viscosity [kg/m/s],  $f$  is the friction factor [8].



**Fig. 5.** Cross section of motor

### 3.3 Analysis of forced convection (Air Gap)

The traditional method to account for heat transfer across air gap is to use the dimensionless convection correlations developed from testing on smooth concentric rotating cylinders by Tayler [8, 14]. In order to judge if the flow in the air gap is laminar, vortex or turbulent, the Taylor number ( $Ta$ ) has to be calculated using Eq. (11). The flow is laminar if  $Ta < 41$ . If  $41 < Ta < 100$  (Our model: 79.5), the flow takes on a vertex form and Eq. (12) has to be used to calculate  $Nu$  [8].

$$Ta = Re \cdot \left( \frac{l_g}{R_r} \right)^{0.5} \quad (11)$$

where  $l_g$  is the air gap radial thickness,  $R_r$  is the rotor outer radius, and  $Re = l_g \cdot v / \mu$

$$Nu = 0.202 \cdot (Ta)^{0.63} \cdot (Pr)^{0.27} \quad (12)$$

### 3.4 Critical parameters

The accuracy of thermal networks is dependent upon several parameters for which reliable data may be difficult to find, for example, the interface gap between components and thermal conductivity data of important materials used in electrical motors [2]. It is difficult to find appropriate thermal data of electrical steel, insulation and impregnation materials. Table 4 shows the thermal conductivity of these materials used in this thermal analysis. It is assumed that electrical steel contains silicon by 1.5% because its value is not released exactly by manufacturer.

**Table 4.** Thermal conductivity of important materials

Material	Thermal Conductivity [W/m°C]	Remarks
Electrical steel	33	Silicon 1.5 %
Epoxy	1.02	Molding of windings
Nomex 410	0.14	Slot liner
Aluminum	168	Housing

The accuracy of a motor's thermal performance prediction is also dependent upon the estimate of the many thermal contact resistances within the machine (stator lamination to housing, slot-liner to lamination, etc) [9]. The data given by Mills [10] can be converted to equivalent air gaps by using a thermal conductivity for air of 0.026 W/m°C. For the typical material interfaces found in electrical machines, values of an interface gap for aluminum – aluminum are in the range 0.0005 to 0.0025 mm and aluminum – iron of 0.0006 to 0.006 mm [10]. The gap between lamination and housing is a function of how well the rough laminated outer surface of the stator is prepared before the housing is fitted. This study used 0.01mm as the gap between lamination and aluminum housing considering the roughness of lamination.

## 4. Thermal analysis and test results

### 4.1 Parameters for thermal analysis

Table 5 and Fig. 6 show the heat transfer coefficients of important parts and thermal resistances by commercial software using theory of chapter 3.

In this study, as the high speed motor is water cooling type, heat transfer coefficients of water jacket channel and interface between stator core and frame are dominant.

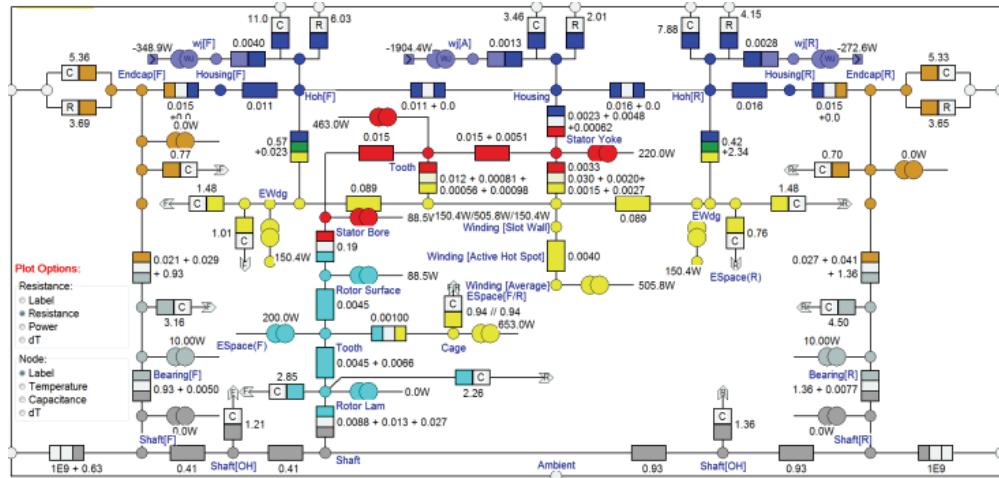


Fig. 6. Thermal equivalent circuit (resistances)

**Table 5.** Heat transfer coefficient of important parts

Components	Heat tran. coef. [W/m <sup>2</sup> /°C]	Remarks
Air gap	128.1	Convection
Water jacket channel	6,997	Convection
From end coil to inner air	133	Convection
From end ring to inner air	210	Convection
From housing to outer air	8.3	Conv.+Rad.
Between stator core and Frame	2,600	Interface gap

#### 4.2 Analysis results and comparison with test results

Fig. 7 is temperature distribution as the thermal analysis results by LPTN method. The details of analysis results are shown in Table 6 which shows analysis results by using loss analysis and test results of chapter 2. Test results of motor temperature are from temperature saturation test of Fig. 8 by using 60kW Dynamo system as shown in Fig. 9.

The difference between stator and rotor temperature is high over 100°C because this model is housing water cooling type (cooling water condition in Table 1). Thermal analysis results using FEM losses are lower than test losses due to lower FEM losses than test losses.

A temperature of rear part of end winding is higher than that of front part because rear part has a little gap with housing as shown in Fig. 5 in assembling process

If the interface gap between housing and lamination or equivalent thermal conductivity between winding and lamination are adjusted, winding temperature in slot can be calculated accurately.

**Table 6.** Temperature distribution by thermal analysis

Method	Winding [°C]			Rotor [°C]		Remarks
	End part (front)	In slot	End part (rear)	core	shaft	
Analysis	71.3	66.5	76.3	172.3	166	By FEM losses
Analysis	77.7	72.2	83.3	185.3	179	By test losses
Test	75.2	65.5	80.4	-	-	

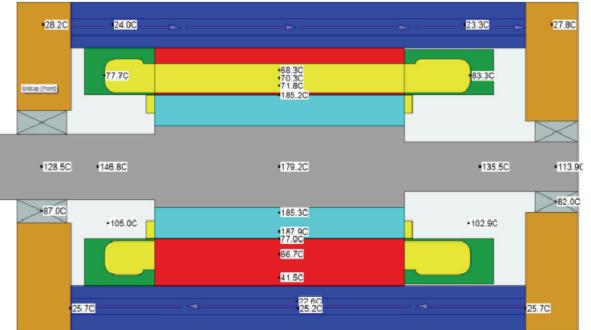


Fig. 7. Temperature distribution (Test losses)

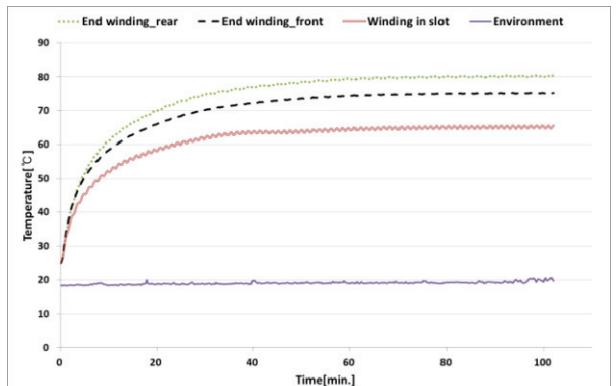


Fig. 9. Dynamo system for testing

## 5. Conclusion

This paper has presented the thermal analysis of water cooling type high speed induction motor by using analytical lumped-circuit method shown in [4]. If critical parameters such as interface gap between materials and thermal conductivity of important materials are selected well considering manufacturing conditions, a temperature of motor can be expected accurately in LPTN method. In motor design step, loss analysis results are also important factors to analyze temperature characteristics.

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