

Novel Techniques for Real Time Computing Critical Clearing Time SIME-B and CCS-B

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Abstract – Real time transient stability assessment mainly depends on real-time prediction. Unfortunately, conventional techniques based on offline analysis are too slow and unreliable in complex power systems. Hence, fast and reliable stability prediction methods and simple stability criterions must be developed for real time purposes. In this paper, two new methods for real time determining critical clearing time based on clustering identification are proposed. This article is covering three main sections: (i) clustering generators and recognizing critical group; (ii) replacing the multi-machine system by a two-machine dynamic equivalent and eventually, to a one-machine-infinite-bus system; (iii) presenting a new method to predict post-fault trajectory and two simple algorithms for calculating critical clearing time, respectively established upon two different transient stability criterions. The performance is expected to figure out critical clearing time within 100ms-150ms and with an acceptable accuracy.

Keywords: Critical clearing time, CCS-B, Clustering identification, Post-fault trajectory, Real time computing, SIME-B, Transient stability

1. Introduction

Power system stability has been recognized as an important problem for secure system operation since the 1920s [1, 2]. Many major blackouts caused by power system instability have illustrated the importance of this phenomenon [3]. Transient instability has been the dominant stability problem on most systems, and has been the focus of much of the industry's attention concerning system stability.

Large-disturbance rotor angle stability or transient stability is concerned with the ability of the power system to maintain synchronism when subjected to a severe disturbance, such as a short circuit on a transmission line. Transient stability depends on both the initial operating state of the system and the severity of the disturbance. Instability is usually in the form of aperiodic angular separation due to insufficient synchronizing torque, manifesting as first swing instability. Instability that may result occurs in the form of increasing angular swings of some generators leading to their loss of synchronism with other generators. The basic factor is how the power outputs of synchronous machines vary as their rotor angles change. If the system is perturbed, the equilibrium between the input mechanical torque and the output electromagnetic torque of each generator is upset, resulting in acceleration or deceleration of the rotors of the machines according to the laws of motion of a rotating body. The power-angle

relationship is highly nonlinear. Beyond a certain limit, an increase in angular separation is accompanied by a decrease in power transfer such that the angular separation is increased further [4].

The dimensions of power network become larger with the development of interconnected local networks. As power systems have evolved through continuing growth in interconnections, use of new technologies and controls, and the increased operation in highly stressed conditions, different forms of system instability have emerged. Hence, a fast and accurate transient stability prediction method is necessary to start up emergency control ensuring security operations of the power grid [5].

There are many transient stability assessment techniques have been proposed such as time-domain (T-D) simulation method [6], transient energy function method [7] and hybrid method [8]. However, contemporary techniques are not fast enough for real time stability assessment. The two methods presented following will fill the need of emergency control in power systems. To reach this objective, the proposed methods have called upon one assumption; that is there are no delays in data collecting from PMUs. This means necessary information is available immediately after the fact.

2. Clustering Identification

Mania Pavella et al. pointed out that, like all One-Machine-Infinity-Bus (OMIB)-based methods, Single Machine Equivalent (SIME) relies on the following two propositions [9]:

Proposition 1: However complex, the mechanism of loss

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of synchronism in a power system originates from the irrevocable separation of its machines into two groups: one composed of the “critical machines” (CMs, subscript C), which are responsible of the loss of synchronism, the other of the “non-critical machines” (NMs, subscript N).

Proposition 2: The stability properties of an OMIB may be inferred from Equal Area Criterion (EAC) constructed for this OMIB.

Thanks for phasor measurement units (PMUs) presence, the dynamic progress of disturbed power systems can be monitor in real time. Measured data from PMUs then are used to analysis to recognize groups of generators with similar characters or coherent generator groups. In [5], a method to define coherent groups is proposed: when $t \in [0, \tau]$, if the relative power angles of generators i and j do not exceed a given criterion $\varepsilon_0 > 0$ at any time, then the two generators can be defined as coherent about the time domain τ , that is

$$\max_{t \in [0, \tau]} |\Delta \delta_i(t) - \Delta \delta_j(t)| \leq \varepsilon_0 \quad (1)$$

Usually ε_0 can be chosen as $5^\circ \leq \varepsilon_0 \leq 10^\circ$ and t , for real time calculation, must be limited then $30\text{ms} \leq t \leq 50\text{ms}$ with the rate of PMUs is 10ms-20ms. But simulations show that at the beginning of the disturbances the relative power angles of various generators are not obvious. To get the coherent information faster and more precisely, the first and second derivative of the power angle with respect to time (angular velocity and angular acceleration) can be used as specimens to calculate their space distances. All the n generators can be grouped into several coherent generator groups [5] by using the following index:

$$d_{ij}(k) = \frac{d_{\omega_{ij}}(k)}{\sum_{k=1}^{n_0} d_{\omega_{ij}}(k)} \quad \text{or} \quad d_{ij}(k) = \frac{d_{\dot{\omega}_{ij}}(k)}{\sum_{k=1}^{n_0} d_{\dot{\omega}_{ij}}(k)} \quad (2)$$

where

$$\begin{aligned} n_0 &= n(n-1)/2 \\ d_{\omega_{ij}}(k) &= \sqrt{\sum_{k=1}^K (\Delta \omega_i(t) - \Delta \omega_j(t))^2} \\ d_{\dot{\omega}_{ij}}(k) &= \sqrt{\sum_{k=1}^K (\Delta \dot{\omega}_i(t) - \Delta \dot{\omega}_j(t))^2} \end{aligned} \quad (3)$$

d_{ij} represents the similarity of output trajectory of various generators, if d_{ij} is less than a certain value ε , it means that the dynamic behaviors of the two generators are similar which can be regarded as a coherent generator group, otherwise they belong to different groups. Following the above procedure, CMs and NMs are identified. However, the main drawback of the grouping generators is the simplicity of the two-machine (or, one-

machine to infinite bus) equivalent which can only approximate an actual multi-machine, multi-bus interconnected electric power system, with lossy components, nonlinearities, high-speed exciters, controllers and stabilizers, regulators, and reactive compensators, having an overall considerable contribution to the transient and dynamic behavior of the system. Since the above contribution is normally beneficial to system stability, the direct methods can only claim a conservative estimation. With the above conditions, any additional simplification introduced in the analysis, or in the computational procedure will further weaken the overall accuracy of the results [12]. Moreover, the performance of grouping generators depends on the value ε .

3. Network Reduction

3.1 The multi-machine model

In the classical simplified model of an n -machine power system, each generator is represented electrically as a constant voltage source behind its transient reactance and each load as constant impedance. The motion of the i^{th} machine is accordingly described by:

$$\dot{\delta}_i = \omega_i; \quad M_i \dot{\omega}_i = P_{mi} - P_{ei} = P_{ai}, \quad i=1, \dots, n \quad (4)$$

where

- δ_i : rotor angle
- ω_i : rotor speed
- M_i : inertia coefficient
- P_{mi} : mechanical input power
- P_{ei} : electrical output power
- P_{ai} : accelerating power

3.2 The equivalent two-machine model

The OMIB transformation results from the decomposition of the system machines into two groups, the aggregation of these latter into their corresponding center of angle (COA). Denoting by $\delta_C(t)$ the COA of the group of CMs, one writes:

$$\delta_C(t) = M_C^{-1} \sum_{k \in C} M_k \delta_k(t) \quad (5)$$

$$\text{Similarly:} \quad \delta_N(t) = M_N^{-1} \sum_{j \in N} M_j \delta_j(t) \quad (6)$$

In the above formulas: $M_C = \sum_{k \in C} M_k$; $M_N = \sum_{j \in N} M_j$

3.3 The equivalent OMIB model

The expressions of corresponding OMIB parameters $\delta, \omega, M, P_m, P_e, P_a$ are derived as follows.

a) Define the rotor angle of the corresponding OMIB by the transformation

$$\delta(t) = \delta_C(t) - \delta_N(t) \quad (7)$$

The corresponding OMIB rotor speed is expressed by

$$\omega(t) = \omega_C(t) - \omega_N(t) \quad (8)$$

where: $\omega_C(t) = M_C^{-1} \sum_{k \in C} M_k \omega_k(t)$; $\omega_N(t) = M_N^{-1} \sum_{j \in N} M_j \omega_j(t)$

a) Define the equivalent OMIB mechanical power by

$$P_m(t) = M \left(M_C^{-1} \sum_{k \in C} P_{mk} - M_N^{-1} \sum_{m \in N} M_{mj} \right) \quad (9)$$

and the resulting OMIB accelerating power by

$$P_a(t) = P_m(t) - P_e(t) \quad (10)$$

In the above expressions, M denotes the equivalent OMIB inertia coefficient

$$M = \frac{M_C M_N}{M_C + M_N}$$

4. Proposed Methods

4.1 SIME-based method (SIME-B)

a) SIME revisited

Literature [9] investigated on SIME. SIME is a transient stability method based on a generalized OMIB. The fundamental difference between the original version of Extended Equal-Area Criterion (EEAC) and SIME is that EEAC relies on a time-invariant OMIB that constructs by assuming the classical simplified machine and network modeling and by “freezing” once and for all the machine rotor angles at t_0 , the initial time of the disturbance inception.

- Equal Area Criterion revisited

EAC relies on the concept of energy. In short, it states that the stability properties of a contingency scenario may be assessed in terms of the stability margin, defined as the excess of the decelerating over the accelerating area of the OMIB $P_a - \delta$ plane:

$$\eta = A_{dec} - A_{acc} \quad (11)$$

- Conditions of unstable OMIB trajectory

An unstable case corresponds to $\eta < 0$, i.e., to

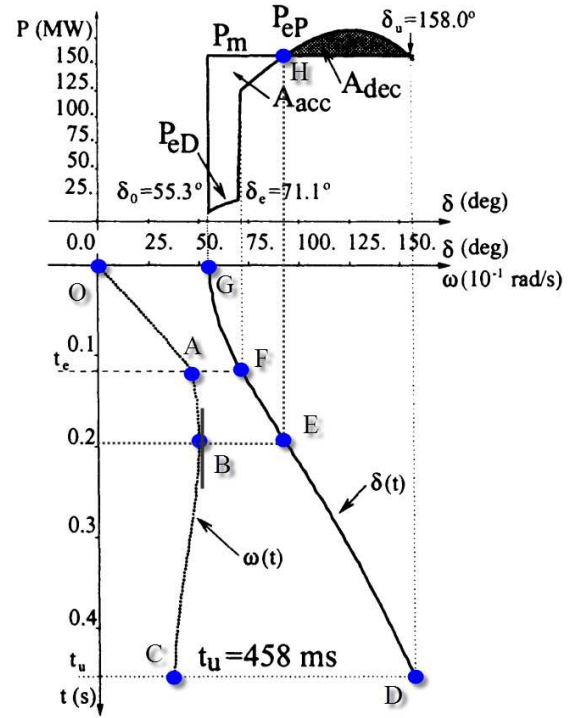


Fig. 1. OMIB $P-\delta$ and T-D representation with $t_e = 117ms$, unstable case

$A_{dec} < A_{acc}$. In such a case, the curve P_{ep} crosses P_m or, equivalently, the accelerating power P_a passes by zero and continues increasing. From a physical point of view, $P_a = 0$ takes place at $\delta = \delta_u$ and marks the OMIB loss of synchronism. An unstable OMIB trajectory reaches the unstable angle δ_u at time t_u as soon as:

$$P_a(t_u) = 0, \quad \dot{P}_a(t_u) = \left. \frac{dP_a}{dt} \right|_{t=t_u} > 0 \quad (12)$$

- Conditions of stable OMIB trajectory

A stable case corresponds to $\eta > 0$, i.e., to $A_{dec} > A_{acc}$. Inspection of Figs. 1 or Fig. 2 suggests that in such a case, the acquired kinetic energy is less than the maximum potential energy: P_{ep} stops its excursion at $\delta = \delta_r$, before crossing P_m . Stated otherwise, at $\delta = \delta_r$, $\omega = 0$ with $P_a < 0$, δ stops increasing then decreases. The above reasoning yields the following statement: a stable OMIB trajectory reaches the return angle δ_r ($\delta_r = \delta_u$) at time t_r as soon as

$$\omega(t_r) = 0, \quad \text{with } P_a(t_r) < 0 \quad (13)$$

b) Proposed method based on SIME (SIME-B)

SIME-B method is developed based on angular velocity prediction and a new transient stability assessment derived from conditions of stable OMIB trajectory. Furthermore, the criterion is made simpler from an interesting observation. Since the new criterion is quite simple,

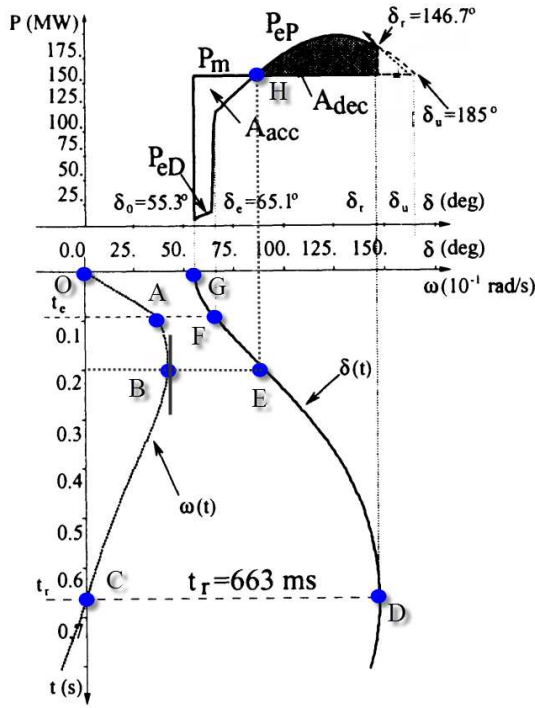


Fig. 2. OMIB $P-\delta$ and T-D representation with $t_e = 92ms$, stable case

computation speed is extremely fast and tallies with real time assessment.

- The angular velocity prediction

Literature [10] first proposed that the generator angular velocity can be obtained by Newton interpolation method and the generator angle in a future time can be calculated by integration of the angular velocity. Since the generator rotor has considerable large inertia, the variation of its angular velocity ω is a smooth procedure.

In order to determine the critical clearing time CTT, the computation method needed be able to predict the post-fault trajectory after fault is cleared. However, that prediction merely based on commencing data directly following fault occurs, normally, contains significant errors. The reasons are the P_a curve is not smooth and clearance time t_e is still a variable. In this paper, a new method on the basis of the angular velocity prediction successfully addresses that problem. Extrapolating angular velocity with a quadratic function of time is a reasonable choice. Hence, ABC curve is of the form:

$$\omega(t) = at^2 + bt + c, \quad a < 0, t \geq t_A \quad (14)$$

and solving for a, b, c based on $\omega(t)$ values taken at three successive time steps. One of these values is angular velocity at point A, $t_A = t_e$, when fault is cleared, and the others at $t_A \pm \Delta t$, $\Delta t = 10ms$. The section OA is nearly linear, therefore, assume that OA is of the form:

$$\omega(t) = kt, \quad t_0 \leq t \leq t_A \quad (15)$$

Obviously, when we move A along the line OA, a, b, c must be different. That is the key of the new method and is able to illustrate how the clearance time affects the following predictions.

Moreover, angle of rotor and angular acceleration are of the forms:

$$\delta(t) = \frac{1}{3}at^3 + \frac{1}{2}bt^2 + ct + d \quad (16)$$

$$\dot{\omega}(t) = 2at + b \quad (17)$$

$$t \geq t_A$$

Once a, b and c are determined and commencing data of angle are available, angle curve $\delta(t)$ and angular acceleration curve $\dot{\omega}(t)$ for $t \geq t_A$ are resolved as well. Taking three points on angular acceleration curve $\dot{\omega}(t)$ and combining with swing equation:

$$M\dot{\omega} = P_a \quad (18)$$

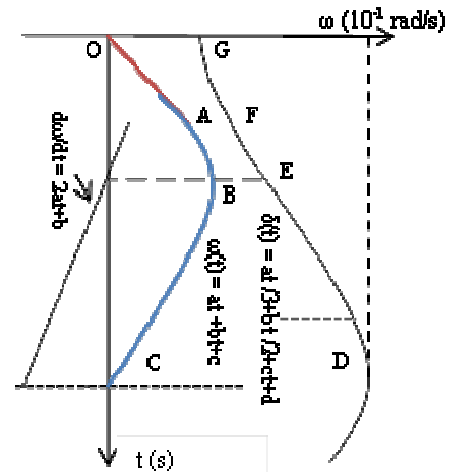


Fig. 3. Post-fault trajectory prediction

corresponding values of P_a will be found. Referring these values of P_a to angle curve $\delta(t)$, P_a curve is defined for $t \geq t_A$:

$$P_a(\delta) = m\delta^2 + p\delta + q, \quad m < 0$$

In other words, m, p and q are solved. Since sufficient information is attained, either EEAC or SIME method can be applied to compute CCT.

- The new transient stability assessment and CCT determining algorithm

Firstly, "angle to instability" δ_u can be computed as:

$$P_a(\delta_u) = m\delta_u^2 + p\delta_u + q = 0 \quad (19)$$

$$\text{Since } \delta_u > \delta_H, \delta_u = \frac{-p - \sqrt{p^2 - 4mq}}{2m} \quad (20)$$

Similarly, from angular velocity curve $\omega(t)$, t_c is found as:

$$\omega(t_c) = at_c^2 + bt_c + c = 0$$

$$\text{Then, } t_c = \frac{-b - \sqrt{b^2 - 4ac}}{2a} \quad (21)$$

The maximum angle in stable case, δ_D , may be calculated as:

$$\delta_D = \frac{1}{3}at_c^3 + \frac{1}{2}bt_c^2 + ct_c + d; \quad t_D = t_c$$

The condition of stable trajectory can be restated as: if $\delta_D < \delta_u$, the system is first-swing stable; if $\delta_D > \delta_u$, the system is unstable. The statement is valid since the system is stable if the maximum angle δ_D is smaller than “angle to instability” δ_u . Based on this claim, a new algorithm to determine CCT is proposed with 4 steps:

- i. Initial clearing time conditions: select a value of t_A for the first location of point A, say $t_A = 100\text{ms}$ after fault occurs
 - Depicting OA segment with coefficient k or building angular velocity curve $\omega(t)$ for $t \leq t_A$
 - Estimating a, b and c to find angular velocity curve, angle of rotor and angular acceleration curve.
 - Predicting P_a and determining m, p, q
- ii. Computing δ_D & δ_u
- iii.
 - If $\delta_D < \delta_u$, increasing or move point A further to point O along OA segment and returning to i.
 - If $\delta_D > \delta_u$, decreasing or move point A closer to point O along OA segment and returning to i.
- iv. Iteration process is complete if in two successive steps, $\delta_D - \delta_u$ changes its sign from positive to negative or vice versa. CTT is determined as average of the last two clearing times.

Remarks:

- The proposed method is faster than the existing SIME due to its simple interpolation and criterions. The difference becomes clearer when the CCT-finding procedures are used repeatedly. The larger number of iterations, the more computationally interesting the proposed method becomes with respect to the existing one

- With respect to the very high non-linear issue, we observe that while the electric power outputs behave in a highly on-linear way particularly at the moment of fault clearance, the angle dynamics and the angle velocity dynamics remain “nice”, which allows for an accurate extrapolation using a quadratic function. This phenomenon

may be a result of physical characteristics of generator rotor having considerable inertia. Since the electric power outputs are complex and depend on clearance time which is still unknown, the predicting steps and the criterions of the proposed method mainly rely on the angle velocity rather than the electric power outputs. This point also distinguishes the proposed methods from the existing SIME in which the electric power is directly predicted and the criterion is defined via the negative margin energy which depends on the electric power.

- Simplifier algorithm

The above algorithm appeals simply to carry out with computers and saves time comparing with the conventional one. Nevertheless, for the sake of real-time computation purposes, it needs to be improved. Notice that in both stable and unstable cases, t_b is almost fixed. The reason is the rotor normally has a large inertia. At point B, angular velocity reaches its maximum, $\omega(t_b) = \omega_{\max}$ and angular acceleration equals zero $\dot{\omega}(t_b) = 0$. E is the point on angle curve such that $t_E = t_b$. From swing equation and $\dot{\omega}(t_b) = 0$, point E is corresponding with point H on P_a curve, where $P_{aH} = P_m - P_e = 0$. Relied on these observations, we can assume that point H and P_a curve are unchanged. Then P_a curve is defined from the first iteration and kept unchanged through the simulation procedure. Finally, computational intensity reduces by a considerable amount.

Admittedly, the simplifier algorithm is based on the simplifying assumption which would affect the accuracy of the results. However, experiments pointed out that the errors are in the acceptable range. This can be explained as an effect of the inertial delay.

4.2 Critical condition for synchronism-based method (CCS-B)

a) Critical condition for synchronism revisited

In [11], Naoto Yorino et al. proposed a critical condition for synchronism for multi-machine systems. As is given in Fig. 4, the critical trajectory converges to unstable

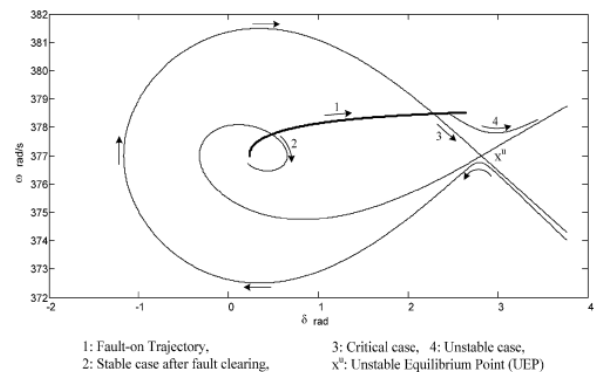


Fig. 4. Trajectories in a phase plane for a single machine to infinite bus system with damping.

equilibrium point (UEP) for a single machine infinite bus system. In this section, the authors propose critical condition for synchronism which suffices in general multi-machine systems. It is known in a single machine case that the synchronizing force disappears when $\frac{\partial P}{\partial \delta} = 0$, where

P is the synchronizing power. A natural extension of the condition for multi-machine case may be written based on singularity condition of synchronizing force coefficient matrix as follows:

$$\left[\frac{\partial P}{\partial \delta} \right] \cdot \nu = 0, \text{ with } \nu \neq 0 \quad (22)$$

where $\nu \in R^n$ is the eigenvector corresponding to zero eigenvalue of matrix $\left[\frac{\partial P}{\partial \delta} \right] \in R^{n \times n}$, and n is the number of generators. The authors further intuitively assume a condition that the eigenvector must agree with change direction of δ . That is, the following equation holds with a scalar $k_s \in R$:

$$\nu = k_s \cdot \dot{\delta} \quad (23)$$

It will be assumed that the above conditions (22) and (23) hold at a point (the end point) on the critical trajectory. Although it is not the complete proof of the stability condition for dynamic system, the equations represent the stationary conditions for the synchronizing power, or as follows:

$$\dot{P} = 0 \quad (24)$$

Since P is basically a function of the rotor angles of generators, the following equation holds:

$$\dot{P} = \left[\frac{\partial P}{\partial \delta} \right] \cdot \dot{\delta} = 0 \quad (25)$$

The conditions have been derived intuitively by the author and will be used without complete proof for instability.

(a) The complete proof of critical condition for synchronism for the two-machine case

The two-machine model is obtained after transforming critical machines group (C) and noncritical machines group (N) into two equivalent machines.

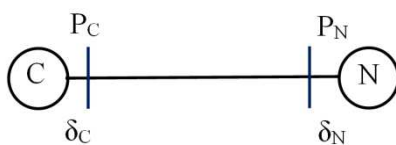


Fig. 5. The two-machine model

where P_C, P_N : electric output of CMs and NMs, respectively

δ_C, δ_N : electric output rotor angle of CMs and NMs, respectively

We define:

$$P = \begin{bmatrix} P_C \\ P_N \end{bmatrix}; \delta = \begin{bmatrix} \delta_C \\ \delta_N \end{bmatrix} \quad (26)$$

$$\left[\frac{\partial P}{\partial \delta} \right] = \begin{bmatrix} \frac{\partial P_C}{\partial \delta_C} & \frac{\partial P_C}{\partial \delta_N} \\ \frac{\partial P_N}{\partial \delta_C} & \frac{\partial P_N}{\partial \delta_N} \end{bmatrix} \quad (27)$$

$$\dot{\delta} = \begin{bmatrix} \dot{\delta}_C \\ \dot{\delta}_N \end{bmatrix} \quad (28)$$

In the vicinity of the end point of the critical trajectory, we can yield:

$$\dot{\delta}_C = k_\delta \cdot \dot{\delta}_N \quad (29)$$

where k_δ is calculated by two successive values of angle. From (25), we have:

$$\dot{P}_C = k_p \frac{\partial P_C}{\partial \delta_C} \quad (30)$$

Swing equation is rewritten as:

$$M_C \ddot{\delta}_C = P_a \quad (31)$$

Taking approximation:

$$M_C \ddot{\delta}_C = \dot{P}_a \Delta \delta_C \quad (32)$$

Since $P_a = P_m - P_e$ and $P_m = const$ then $\dot{P}_a = \dot{P}_e = \dot{P}_C = h$; and $\Delta \delta_C = \delta_C(t) - \delta_C(t=0) = \delta_C - const$ then $\ddot{\delta}_C = \Delta \ddot{\delta}_C$, we have:

$$M_C \cdot \Delta \ddot{\delta}_C = h \cdot \Delta \delta_C \quad (33)$$

If $h < 0$, (33) has real roots of the form:

$$\Delta \delta_C(t) = a_1 e^{\omega_n t} + a_2 e^{-\omega_n t} \quad (34)$$

where $\omega_n = \sqrt{\frac{-h}{M_C}}$

The part of $a_1 e^{\omega_n t}$ will go to infinity or the system will be unstable.

Then $h=0$ is the boundary of stability.

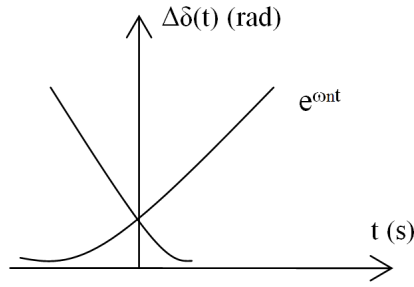


Fig. 6. Unstable case with a negative h

c) The proposed CCS-B method

CCS-B method is alternative one in this paper, which basically relied on behavior of the coefficient $\frac{dP_C}{d\delta_c} = h$.

When h changes its sign from positive to negative, the system loses synchronism. The main difference between CCS-B method and SIME-B is that in CCS-B, the system is only reduced to a two-machine model. Therefore, the result is more accuracy. Similar to SIME-B, the CCS-B algorithm relies on the 4 following steps:

- i. Initial clearing time conditions: select a value of t_A for the first location of point A, say $t_A = 100\text{ms}$ after fault occurs
 - Depicting OA segment with coefficient k or building angular velocity curve $\omega(t)$ for $t \leq t_A$
 - Estimating a, b and c to find angular velocity curve, angle of rotor and angular acceleration curve.

- Predicting $\frac{dP_C}{d\delta_c}$

- ii. Computing δ_D & δ_{cri} , where δ_{cri} satisfies

$$\frac{dP_C}{d\delta_{cri}} = 0$$

- iii.
 - If $\delta_D < \delta_{cri}$, increasing or move point A further to point O along OA segment and returning to i.
 - If $\delta_D > \delta_{cri}$, decreasing or move point A closer to point O along OA segment and returning to i.

- iv. Iteration process is complete if in two successive steps, $\delta_D - \delta_{cri}$ changes its sign from positive to negative or vice versa. CTT is determined as average of the last two clearing times.

Notice that OMIB model in SIME-B is replaced with CMs model in CSS-B method.

5. Case Study

5.1 Numerical examination for SIME-B method

In order to illustrate the effectiveness of SIME-B Method, we have carried out numerical examinations using

10-machine, 39-bus New England Test System. The simulation platform used for the proposed approach is the PC with Intel(R), Core(TM)-2 CPU-2.13 GHz with 4 GB of RAM.

Comparing with conventional time domain simulation method, SIME-B method has shown a good performance with small errors around 0.002s-0.005s while consuming time is extremely fast, less than 105ms. For SIME-B simplifier method, the errors are around 0.001s-0.007s and the consuming time is less than 93ms. The estimated CCTs for various fault location have listed in Table 1 below:

Table 1. CCTs of 39-Bus new england test system

Fault line	CCT by T-D method (10 ⁻³ s)	CCT by SIME-B (10 ⁻³ s)		Computing time of SIME-B (10 ⁻³ s)	
		Full	Simplifier	Full	Simplifier
2-3	263	268	270	90	80
4-14	256	261	259	102	93
6-11	237	235	233	105	92
15-16	241	244	240	87	80
23-24	216	213	218	101	92
25-26	216	216	215	97	81

5.2 Test results of CCS-B method

CCS-B method has been tested on IEEE 7-machine, 57-bus system. The results pointed out precise values of CTT with deviations around 0.001s and execution time is less than 101ms. The results are more accuracy than SIME-B method since the system is only reduced to a two-machine system. Computation results are shown in the following table:

Table 2. CCTs of IEEE 57-Bus system

Fault line	CCT by T-D method (10 ⁻³ s)	CCT by (10 ⁻³ s)		Computing time (10 ⁻³ s)	
		CCS-B	SIME-B (Full)	CCS-B	SIME-B (Full)
2-3	171	170	168	86	89
3-4	241	241	244	92	95
3-15	237	236	234	80	85
4-5	306	306	309	101	99
6-5	183	184	186	100	98
6-8	182	183	182	82	85
7-8	189	189	188	85	87

Table 3. CCTs of IEEE 118-Bus system

Fault line	CCT by T-D method (10 ⁻³ s)	CCT by (10 ⁻³ s)		Computing time (10 ⁻³ s)	
		CCS-B	SIME-B (Simplifier)	CCS-B	SIME-B (Simplifier)
1-3	825	820	834	135	136
15-33	670	673	678	129	130
37-39	586	592	572	126	126
100-103	270	277	261	124	120
105-107	761	755	758	132	119
104-105	1005	1010	1015	131	136

When the systems become larger, much time for handling the big data and running the T-D program will

slow down the CCT determining process. Parallel computing in case of centralized control scheme or distributed computing in case of distributed control scheme should be applied in order to reduce the computing time needed and make the proposed methods becomes even more attractive to real-time applications.

6. Conclusion

This paper proposed two new methods for transient stability assessment: SIME-B and CCS-B with introduction of PMUs. CCT is determined in extremely short time, obviously, is eligible for real time analysis. The new model applied for predicting post-fault trajectory based on angular velocity prediction successfully overcomes problem of accelerating power estimation, which is unsmooth. SIME-B method can be executed even faster since the time angular velocity reaches the maximum, t_B , is fixed. However, in some cases, a large deviation of P_a will lead to inaccurate results. On the other hand, CCS-B seems to be a promising method because of its precision and rapid convergence.

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