

# 실험계획법에 의한 차량 현가계의 진동저감에 관한 연구

## A Study on the Vibration Reduction of Vehicle Suspension system Using Design of Experiment

박진수\* · 정운창\* · 정재은\* · 이정윤\*\* · 오재응†  
**Jin-Su Park, Un-Chang Jeong, Jae-Eun Jeong, Jung-Youn Lee and Jae-Eung Oh**

1.

2.

가

S/N Ratio(Signal to noise ratio)

가

(Table of orthogonal arrays)

가

S/N Ratio

가

가

가

가

가

4

4 /

(multi-dimensional spectral analysis ;

MDSA)

가

(Taguchi method)

가

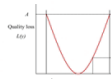
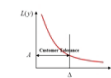
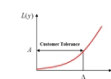
	Loss Function	S/N Ratio
원목특성	$L(y) = k \left[ \frac{1}{n} \sum_{i=1}^n (y_i - m)^2 \right]$ 	$SN = 10 \log \left[ \frac{\frac{1}{n} (S_m - V)}{V} \right]$
망대특성	$L(y) = k \left[ \frac{1}{n} \sum_{i=1}^n \frac{1}{y_i^2} \right]$ 	$SN = -10 \log \left[ \frac{\frac{1}{n} \sum_{i=1}^n \frac{1}{y_i^2}}{\frac{1}{n} \sum_{i=1}^n \frac{1}{y_i^2}} \right]$
망소특성	$L(y) = k \left[ \frac{1}{n} \sum_{i=1}^n y_i^2 \right]$ 	$SN = -10 \log \left[ \frac{\frac{1}{n} \sum_{i=1}^n y_i^2}{\frac{1}{n} \sum_{i=1}^n y_i^2} \right]$

Fig.1 The Formula of Loss Function and S/N Ratio

3.

2000cc

(B&K Type 4189)

가

†

E-mail : jeoh@hanyang.ac.kr

Tel : (02) 2220-0452, Fax : (02)2299-3153

\*

\*\*

(PCB Type 356)  
60kph  
3dB  
3 가 , ,  
가 가 가  
4. 가

Fig.3 , 500Hz  
Peak 221Hz

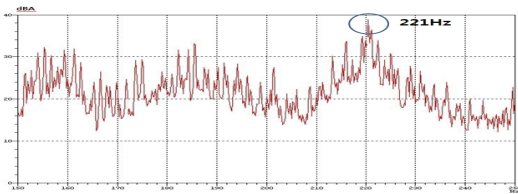


Fig.3 A-weighted SPL of the driver's ear position under 60kph

( )  
( )  
(PCF) Table1  
(MCF) 0.998  
가

Table 1 The partial coherence function among input and output signals

Frq.(Hz)	$\gamma_{1y}^2$	$\gamma_{2y-1}^2$	$\gamma_{3y-1,2}^2$	$\gamma_{4y-1,2,3}^2$
221Hz	0.949	0.376	0.837	0.526

5. 가

가

가

ABAQUS

가 가 가  
가 가  
가 가  
(dB)  
bead (A),  
(B) (C), bush guide  
(D)  
3  
가 4 3

Table 2 L9

Table 2 L9 Orthogonal array

No	A	B	C	D
1	1	1	1	1
2	1	2	2	2
3	1	3	3	3
4	2	1	2	3
5	2	2	3	1
6	2	3	1	2
7	3	1	3	2
8	3	2	1	3
9	3	3	2	1

Table 3 Estimation of the optimal design through FE Analysis

Frq.(Hz)	Original Design	Optimal Design	Difference
221Hz	49.8dB	41.4dB	-8.4dB

5.

가

8.4dB(16.9%)

221Hz