

STP Extended Top 에 설치되는 Pull-In Winch 하부구조 보강방법

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Application Method for the Structures
Under the Full-In Winch on STP Extended Top

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Abstract

The latest trend of development for submarine oil field is caused by the drained oil and gas for ground field, and FPSO is a keyword as the development for submarine oil field. FPSO (Floating Production Storage Offloading) is a kind of vessel type have a topside system for production of oil and gas and store them until the oil or gas can be offloaded onto a tanker or transported through a pipeline.

Prior to the introduction of reinforcement under the pull-in winch on the STP extended top as the object of this paper, the technical background shall be introduced such as FPSO and the system and main equipments for STP as follows.

The original structural concept for reinforcement of pull-in winch on the STP top and extended structure on moonpool was proposed by buyer's engineering team but it was much modified and improved in accordance with builder's fabrication and construction method.

※Keywords: Floating production storage and offloading, Submerged turret production system

1. 서론

본 논문에서는 FPSO 에 대한 개략적 설명과 함

께 초도 제품 STP system 에 대한 일반적인 개념을 소개 하고자 한다.

FPSO production system 중 하나인 STP (Submerged Turret Production system) 핵심 장비인 Pull-in Winch 하부 보강에 대해서 구조해석

및 Buckling 해석 결과를 토대로 실선 FPSO 에 적용 하였다.

Pull-in Winch 하부 보강 검토 시 고려해야 할 가장 중요한 요소인 Loading condition 과 Explosion pressure 는 선주 및 maker 에서 제공한 정보를 적용 했으며 하부 보강 방법 및 Scantling 적용은 선급 rule 및 Maker 요구 사항을 고려한 최적 설계를 구현 하였다

2. 본론

2.1 FPSO 및 주요 System 개요

- FPSO: 부유식 원유생산저장 설비 “ Floating Production Storage Offloading vessel”

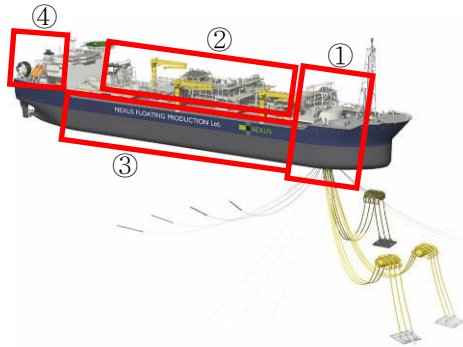


Fig. 1 FPSO with STP

- ① Turret & Mooring System (STP): 유정으로 부터 oil 을 loading 하는 동시에 선박을 해저에 계류하는 system
- ② Topside Processing System: 해저에서 채굴된 유정 혼합물(물+가스+자갈+모래+불순물+oil...)로부터 필요한 oil 및 gas 를 수송 가능한 상태로 분리하는 정제 system
- ③ Cargo Tank System: Topside Processing System 에서 분리된 정제유를 임시로 저장하는 tank system
- ④ Offloading System(SDS): Cargo Tank System 에 저장된 정제유(crude oil)를 Crude oil tanker 로 off loading 시키는 System 즉, FPSO 에 저장된 원유를 직접 수출선인 COT 에 선적하는 장치임.

2.2 STP System 소개

- STP(Submerged Turret Production System): Drill ship 에 의해서 탐사된 유정 field 해상에 설치된 Buoy 와 FPSO 에 설치된 STP 를 연결하면 압력 차에 의하여 유정의 oil 이 Riser(유정 pipeline) 와 swivel stack 을 거쳐 topside 로 loading 되며 동시에 Buoy 에 설치된 Mooring line 에 의해서 FPSO 를 Operation field 에 계선시키는 역할을 동시에 수행 한다.

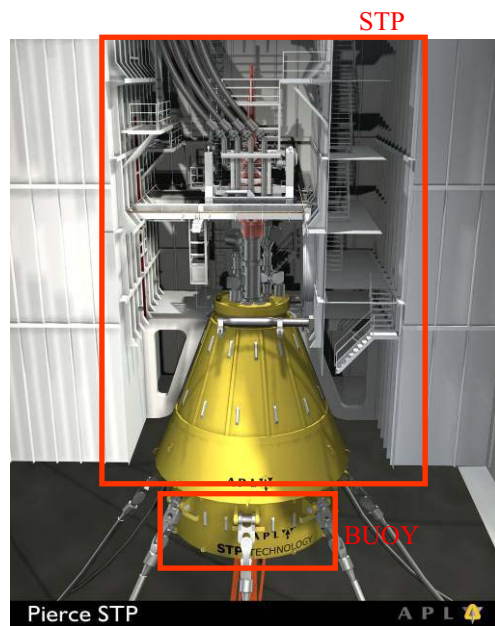


Fig. 2 Image for Buoy and STP with hull

2.3 STP 주요 기기

- **Buoy**: Operation field 에서 해저의 모든 Riser lines(해저의 기름을 FPSO 로 Loading 시키는 pipe lines)과 Mooring lines 을 일체화하여 해상에 Buoy 로서 떠 있으며 본선 FPSO 가 Field 에 도착하면 STP 와 결합되어 Oil loading 및 Mooring 역할을 수행 하게 된다. 황천 시 본선 FPSO 와 Buoy 를 Dis-connect 함으로서 FPSO 를 대피 시킬 수 있으며 황천이 끝난 후 다시 Connection 시킬 수 있도록 설계 되어 있다.



Fig. 3 Picture for translation of buoy

- **MCM(Mating Cone Module)**: 해상에 떠있는 Buoy 와 본선 FPSO 를 연결하는 Hull structure 로서 선체 구조화하는 구조물임. Buoy 를 통해서 전달되는 본선 FPSO 의 모든 힘(풍력/조력/태풍...)을 MCM 이 지탱 해야 하기 때문에 반드시 선체구조 해석을 통해서 강도가 검증 되어야 한다.



Fig. 5 Picture for construction of MCM

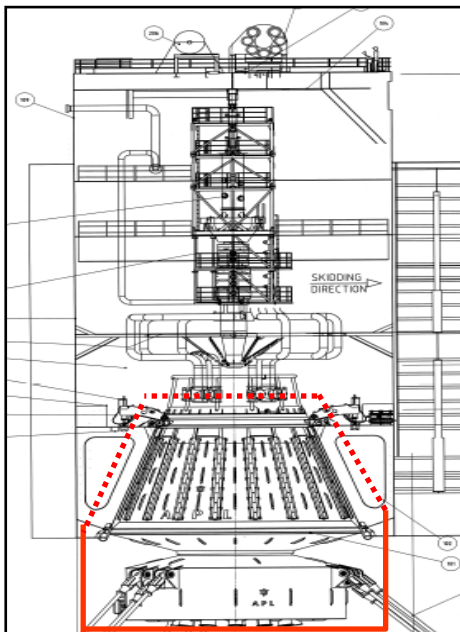


Fig. 4 Display the buoy on STP arr't

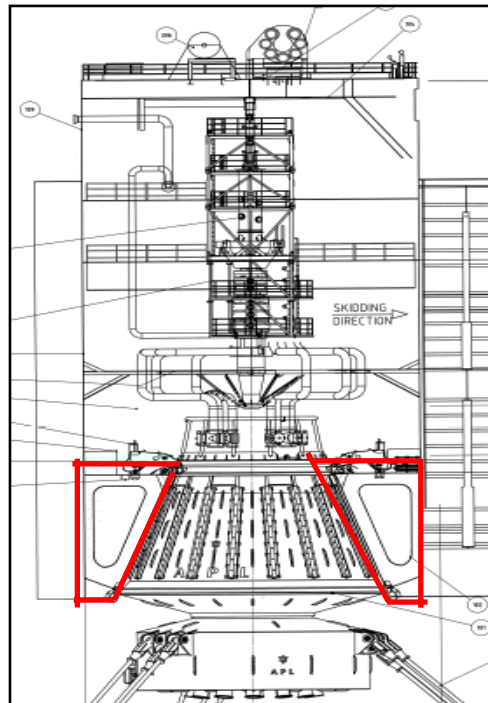


Fig. 6 Display the MCM on STP arr't

- **Locking device:** Operation field 상에서 buoy 를 14 개의 Locking mechanism 으로 체결하여 Mating cone module 에 고정하는 역할을 수행.



Fig. 7 Picture for the locking device in the shop

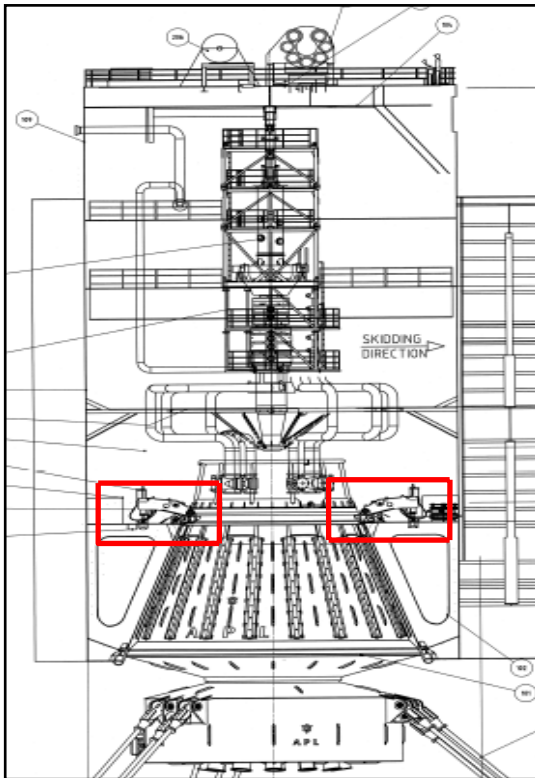


Fig. 8 Display the locking device on STP arr't

- **Shipboard piping system:** Buoy 의 riser 을 통하여 loading 된 유정 혼합물이 2 개의 piping system 을 거쳐 Topside 의 Processing system 으로 이송.

Topside 에서 정제된 후 걸러진 water 와 gas 는 3 개의 별도 piping system 을 거쳐 유정으로 재 투입되어 유정의 압력을 높임으로 유정의 oil 이 원활하게 topside 로 loading 되게 함.



Fig. 9 Picture for the pipes before installation

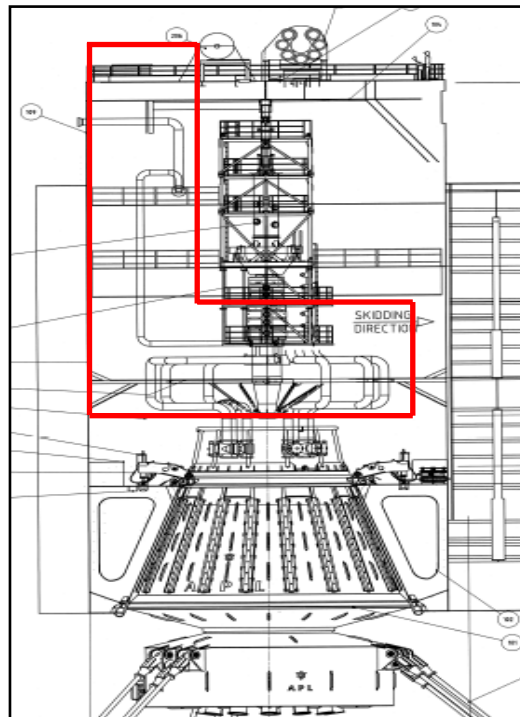


Fig. 10 Shipboard piping system on STP arr't

- **Swivel stack:** Riser 와 STP deck 상단의 piping 을 거쳐 loading 된 유정혼합물을 1 차 정제한 후 shipboard piping system 으로 다시 전달하는 System.

- **Trolley beam:** Swivel stack 을 포함한 Trolley structure 를 선수/선미간 방향으로 이동할 수 있도록 하는 Beam 구조물(sliding 부분을 stainless steel 로 처리함).



Fig. 11 Picture for Swivel stack after installation

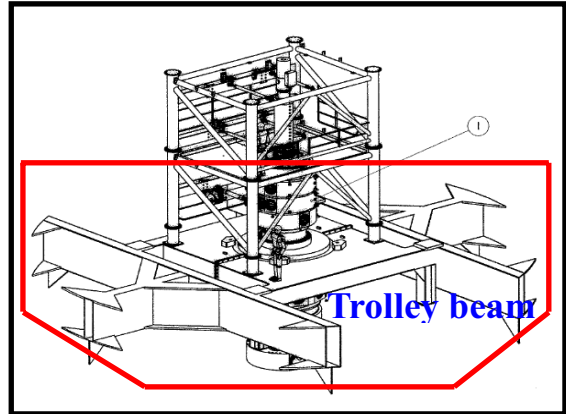


Fig. 13 Image for Trolley beam with swivel stack

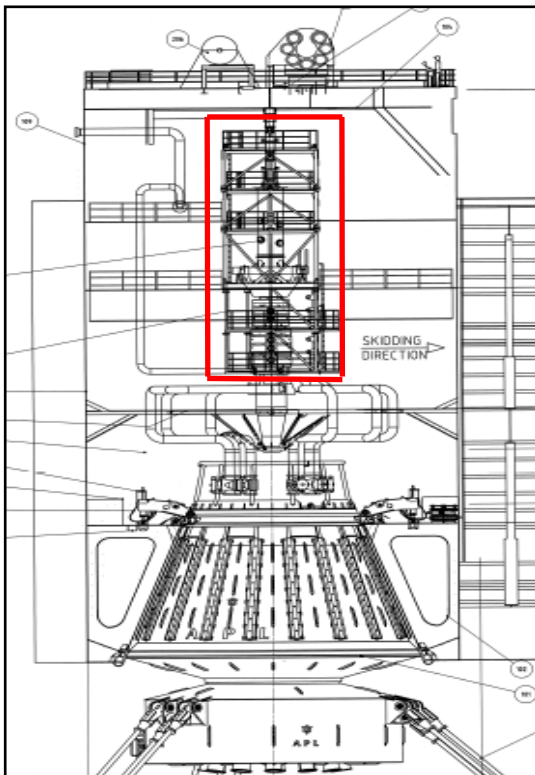


Fig. 12 Display the swivel stack on STP arr't

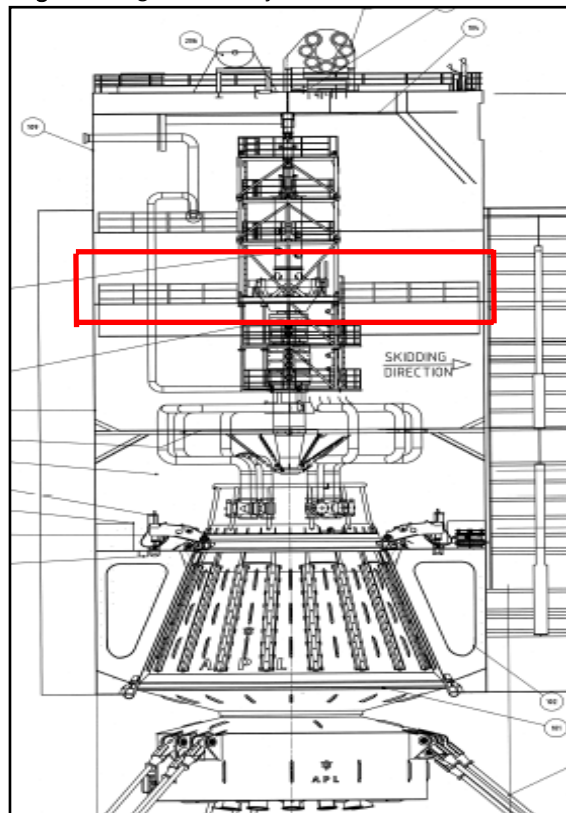


Fig. 14 Display Trolley beam on STP arr't

- Pull-in winch & Spooling device:

Pull-in winch: field 에 위치한 buoy 를 Towing 하여 Mating cone module 에 체결하기 위한 동력을 전달하는 winch
 Spooling device: buoy Towing 용 rope 를 원활하게 Drum 에 감을 수 있도록 guide 하는 system.



Fig. 15 Picture for Pull-in winch in the shop



Fig. 16 Picture for spooling device before installation on STP top

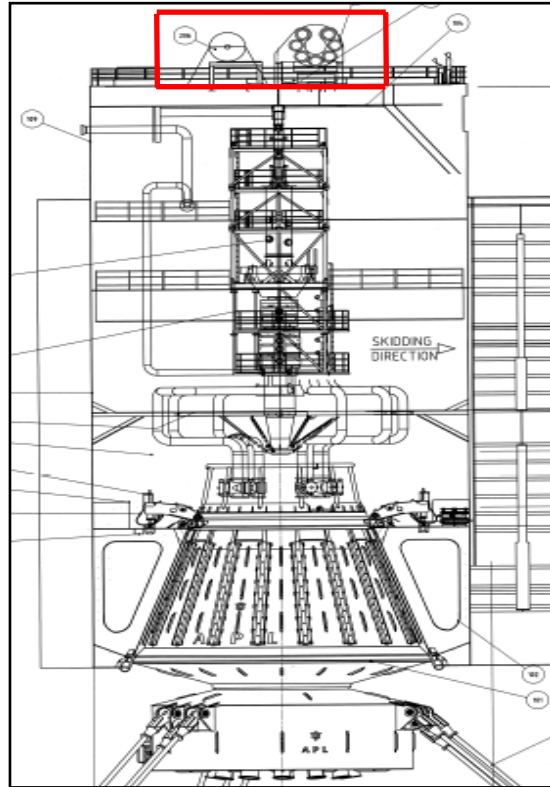


Fig. 17 Display Pull-in winch & spooling device on STP arr't

- Blast Panel: Explosion 에 대비한 완충 역할을 수행, 각 panel 끝단에 hinge 을 체결, 폭발시 deck damage 최소화

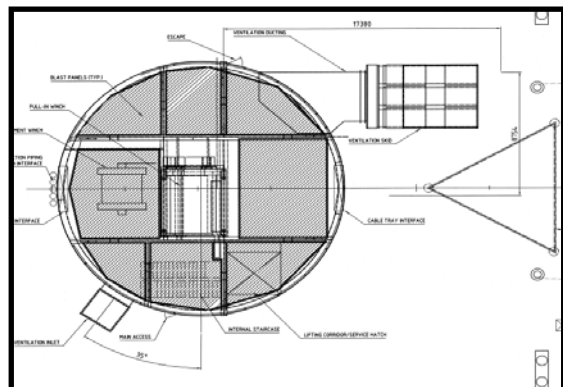


Fig. 18 Picture for blast panel

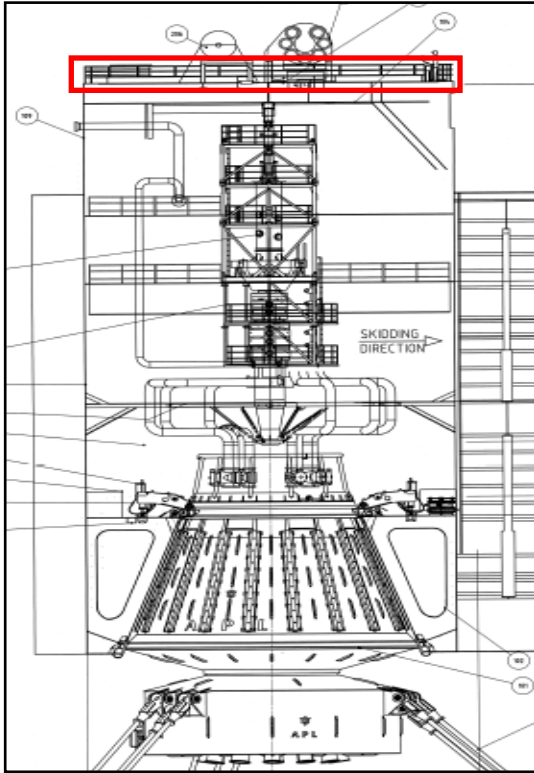
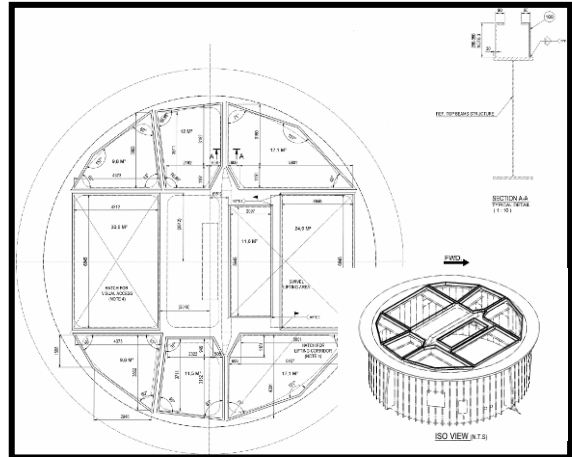
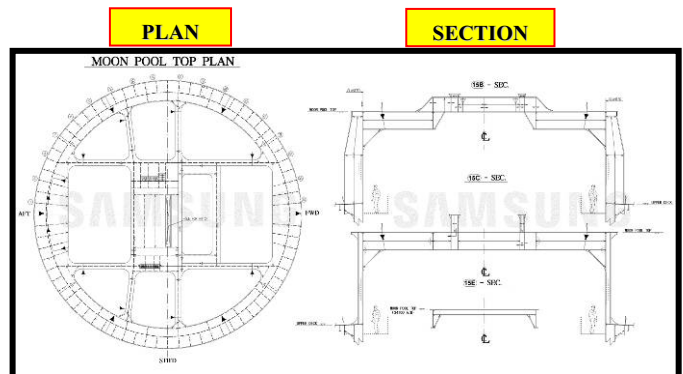


Fig. 19 Display Blast panel on STP arr't

2.4 Structure Analysis for the supporting structure under the Pull-in Winch on STP top

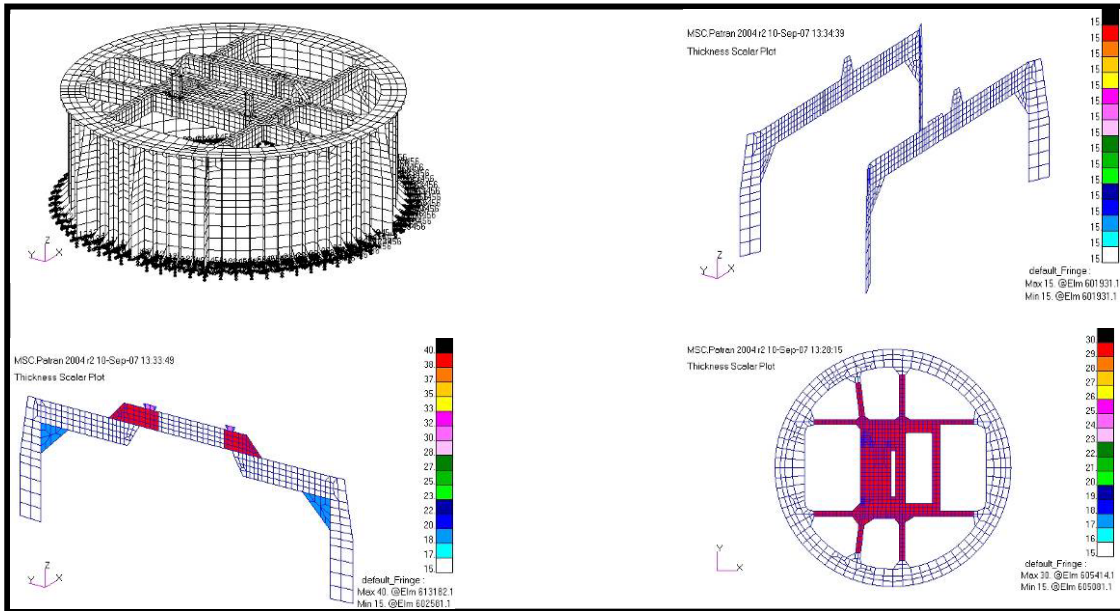


2.4.1 Design of STP Pull-In winch supporting structures.

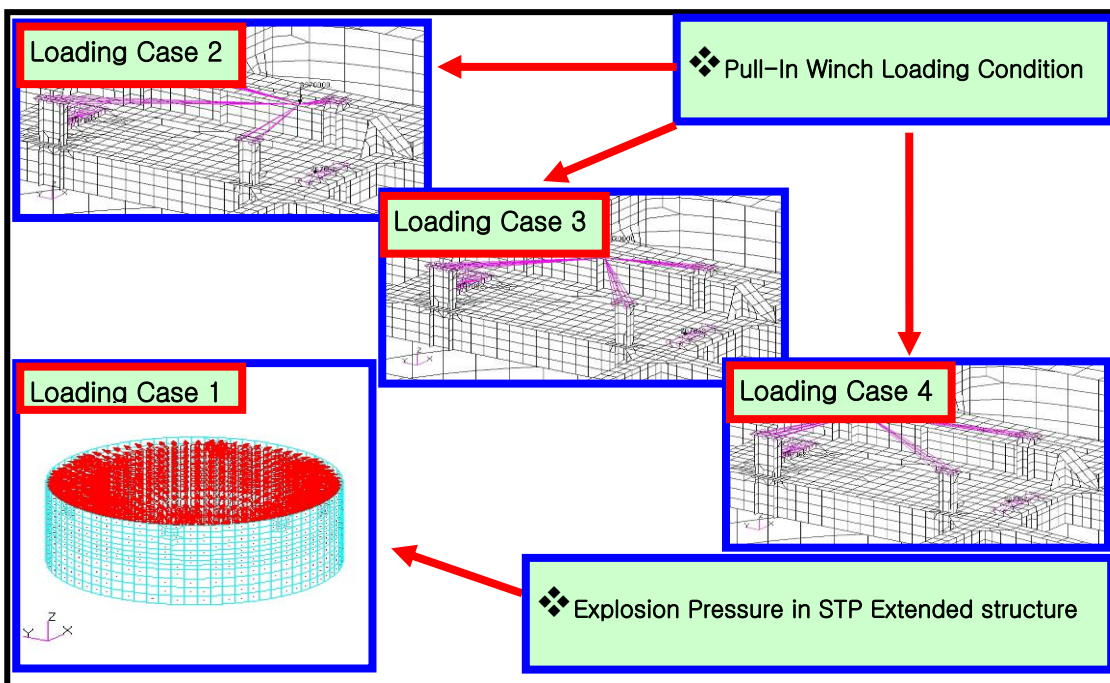


- ❖ Extended Structure for STP 에 대한 선주 proposal(Horizontal system) 검토 후 강도 및 공법 고려한 최적설계(Vertical system)적용 및 선주/급 승인완료.
- ❖ 구조 변경에 따른 Interface 항목들(Pull-In Winch/ Spooling Device/ Explosion Panel)의 선주/Maker/Yard 상호 F-back 및 Confirm.

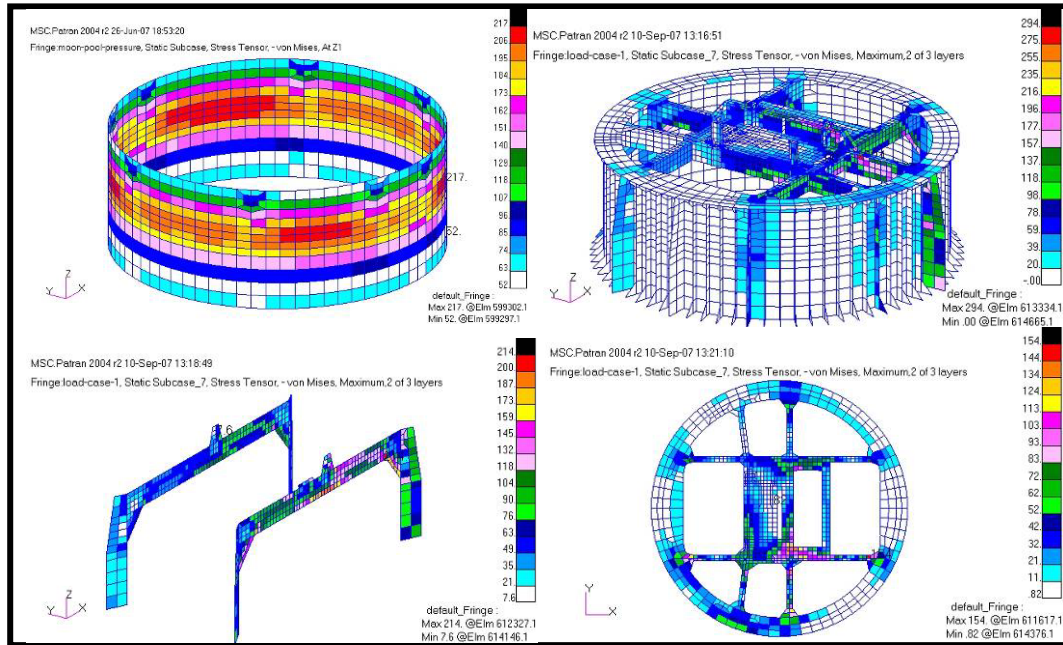
2.4.2 3D Modeling



2.4.3 Loading Cases



2.4.4 F.E Model & Analysis




2.4.5 F.E Analysis Result & Bucking Check

F.E Analysis Result

Loading Cases	VonMises	Allowable	Remarks
Maximum stress for	217	308	OK
Maximum stress for	294	308	OK
Maximum stress for	226	308	OK
Maximum stress for	266	308	OK

Buckling Check for Explosion & Pull-In Winch

	Recommended Practice DNV-RP-C202 October 2002	BUKLING STRENGTH OF SHELLS Version 9.95 Hull, April 2006 Ship Id: 12345	Sign: Time: 19:11 Date: 07.06.27																																																																			
	<p>PRINCIPAL DATA</p> <p>Identification: _____</p> <p>Modulus of Elasticity $E = 210000$ N/mm²</p> <p>Material Factor $\gamma_f = 1.000$ -</p> <p>Poissons Ratio $\nu = 0.3$</p> <p>Shear Modulus $G = 80769$ N/mm²</p> <p>SHELL Longitudinal Stiffened shell (Force input)</p> <p>Thickness $t = 12.0$ mm</p> <p>Yield stress $f_y = 355.0$ N/mm²</p> <p>Shell Radius (middle of plate) $r = 8500.0$ mm</p> <p>Distance between rings $l = 3450.00$ mm</p> <p>Length of shell $L = 6900.0$ mm</p> <p>Eff. Buckling length factor $k = 1.0$ -</p> <p>LONGITUDINAL STIFFENER L-profile</p> <p>Stiffener type</p> <p>Yield stress $f_y = 355.0$ N/mm²</p> <p>Total Height of section $h_{tot} = 350.0$ mm</p> <p>Web thickness $t_w = 12.0$ mm</p> <p>Flange width $b_{fe} = 100.0$ mm</p> <p>Flange thickness $t_{fe} = 17.0$ mm</p> <p>Stiffener spacing $s = 740.0$ mm</p> <p>RING STIFFENER AND RING FRAMES</p> <table border="0"> <tr> <td colspan="2">Ring stiffener:</td> <td colspan="2">Heavy Ring Frame:</td> </tr> <tr> <td>Stiffener Type</td> <td></td> <td>Stiffener Type</td> <td></td> </tr> <tr> <td>Yield stress $f_y =$</td> <td>N/mm²</td> <td>Yield stress $f_y =$</td> <td>N/mm²</td> </tr> <tr> <td>Total Height of section $h_{tot} =$</td> <td>mm</td> <td>Total Height of section $h_{tot} =$</td> <td>mm</td> </tr> <tr> <td>Web Thickness $t_w =$</td> <td>mm</td> <td>Web Thickness $t_w =$</td> <td>mm</td> </tr> <tr> <td>Flange width $b =$</td> <td>mm</td> <td>Flange width $b =$</td> <td>mm</td> </tr> <tr> <td>Flange thickness $t_f =$</td> <td>mm</td> <td>Flange thickness $t_f =$</td> <td>mm</td> </tr> <tr> <td>Max flange outstand $b_{f'} =$</td> <td>mm</td> <td>Max flange outstand $b_{f'} =$</td> <td>mm</td> </tr> <tr> <td>Distance between tripping brackets $l_T =$</td> <td>mm (use 0 if none)</td> <td>Distance between tripping brackets $l_T =$</td> <td>mm</td> </tr> <tr> <td>Initial out of roundness of stiffene: $\delta_o = 0.005 * r$</td> <td></td> <td>L. betw. Girders: $L_H =$</td> <td>mm</td> </tr> </table> <p>Fabrication method: _____</p> <p>LOAD DATA:</p> <p><input checked="" type="radio"/> Specify Forces</p> <table border="0"> <tr> <td>Shell</td> <td></td> <td>Panel</td> <td></td> </tr> <tr> <td>Design Axial force, $N_{sd} = 20000.00$ kN</td> <td></td> <td>Design axial stress, $\sigma_{a,sd} = 19.01$ N/mm²</td> <td></td> </tr> <tr> <td>Design bending mom. $M_{sd} = 0.00$ kNm</td> <td></td> <td>Design bending stress, $\sigma_{m,sd} = 0.00$ N/mm²</td> <td></td> </tr> <tr> <td>Design torsional mom. $T_{sd} = 0.00$ kNm</td> <td></td> <td>Design torsional stress, $\tau_{t,sd} = 0.00$ N/mm²</td> <td></td> </tr> <tr> <td>Design shear force, $Q_{sd} = 0.00$ kN</td> <td></td> <td>Design shear stress, $\tau_{Q,sd} = 0.00$ N/mm³</td> <td></td> </tr> <tr> <td>Design lateral pressure, $p_{sd} = 0.300$ N/mm²</td> <td></td> <td>Design lateral pressure, $p_{sd} = 0.300$ N/mm²</td> <td></td> </tr> <tr> <td></td> <td></td> <td>Additional hoop stress, $\sigma_{h,sd} = 0.00$ N/mm²</td> <td></td> </tr> </table>			Ring stiffener:		Heavy Ring Frame:		Stiffener Type		Stiffener Type		Yield stress $f_y =$	N/mm ²	Yield stress $f_y =$	N/mm ²	Total Height of section $h_{tot} =$	mm	Total Height of section $h_{tot} =$	mm	Web Thickness $t_w =$	mm	Web Thickness $t_w =$	mm	Flange width $b =$	mm	Flange width $b =$	mm	Flange thickness $t_f =$	mm	Flange thickness $t_f =$	mm	Max flange outstand $b_{f'} =$	mm	Max flange outstand $b_{f'} =$	mm	Distance between tripping brackets $l_T =$	mm (use 0 if none)	Distance between tripping brackets $l_T =$	mm	Initial out of roundness of stiffene: $\delta_o = 0.005 * r$		L. betw. Girders: $L_H =$	mm	Shell		Panel		Design Axial force, $N_{sd} = 20000.00$ kN		Design axial stress, $\sigma_{a,sd} = 19.01$ N/mm ²		Design bending mom. $M_{sd} = 0.00$ kNm		Design bending stress, $\sigma_{m,sd} = 0.00$ N/mm ²		Design torsional mom. $T_{sd} = 0.00$ kNm		Design torsional stress, $\tau_{t,sd} = 0.00$ N/mm ²		Design shear force, $Q_{sd} = 0.00$ kN		Design shear stress, $\tau_{Q,sd} = 0.00$ N/mm ³		Design lateral pressure, $p_{sd} = 0.300$ N/mm ²		Design lateral pressure, $p_{sd} = 0.300$ N/mm ²				Additional hoop stress, $\sigma_{h,sd} = 0.00$ N/mm ²
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3. 결론

FPSO STP system 적용관련 각종 system 에 대한 전반적인 구조 및 기능 이해와 system 상 최대 hull loading 이 작용하는 최적의 Pull-in Winch 하부 보강구조 설계를 위한 중요한 요소들을 살펴보았다.

하기와 같이 설계 시 고려 해야 하는 중요한 사항을 기술 하였다.

STP Pull-In Winch 하부 보강 설계 고려조건

- ☞ Interface Joint 정보 접수/검토/협의
- ☞ 선체/의장/생산공정 유기적 협의 진행 필요
- ☞ 각종 의장 품 간섭 회피 및 Maintenance 고려
- ☞ 제작 작업 성 및 용접성을 고려한 부재 배치
- ☞ Load/Loading Condition 를 고려한 Strength 및 Fatigue 해석을 통한 전반적인 구조 안정성 검토



< 김 태 욱 >

< 이 명 실 >

< 오 태 만 >